

Ch 10 Rotational Motion

Translational motion

- Bowling ball **sliding** on icy frictionless surface



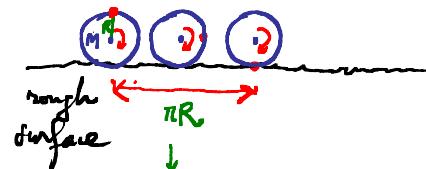
Translation: distance travelled
doesn't depend on size



- Sliding balls of equal masses but different radii ($R_2 = \frac{R}{2}$) follow same translational motion, since here they can be considered as point-like objects of mass M. **size doesn't matter**
- In this translational motion there is no rotation, orientation of ball (red dot) stays the same

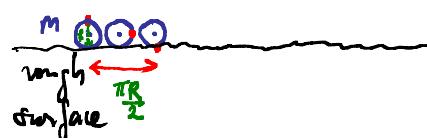
Rotational motion

- Bowling ball **rolling** on rough surface



Rolling motion: distance travelled depends on size R

- Bowling ball of half radius: $R_2 = \frac{R}{2}$



- In rolling motion - with rotation around CM, **size does matter!**

- Rolling motion** \rightarrow Rotation w.r.t. CM (red dot position changes)
 \rightarrow As a consequence of this rotation: translation of CM (TIR: in the diagram above)
- Pure rotation motion without any translation of CM



- Examples: i) car wheels
- | |
|---|
| icy road: translational motion
road under normal conditions: rolling motion
in sand: pure rotational motion |
|---|

ABS in cars : Anti-locking Braking System:

allows wheels to rotate as we apply brakes

No ABS

Apply brakes \rightarrow wheels are blocked
 \rightarrow car slide forward, no wheel rotation & wheels

KE can only go into translational motion

ABS

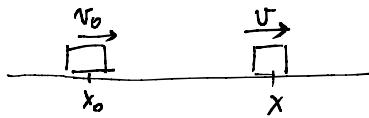
Apply brakes \rightarrow wheels still rotate in rolling motion to a stop.

KE goes into $\left\{ \begin{array}{l} \text{Translation} \\ + \\ \text{Rotation of 4-wheels} \end{array} \right\}$ $\left\{ \begin{array}{l} \text{Shorten} \\ \text{distance} \end{array} \right\}$ dropping distance !

Quantitative Descriptions

Translational motion

- Change of position
- Variables: { position x
velocity v
acceleration a



$$\bar{v} = \frac{v_0 + v}{2} \quad (\text{linear})$$

average velocity

$$\bar{v} = \frac{\Delta x}{\Delta t} = \frac{x - x_0}{\Delta t}$$

$$\bar{\omega} = \frac{\omega_0 + \omega}{2} \quad (\text{angular})$$

$$v = \frac{dx}{dt} \quad \left(\frac{m}{s} \right)$$

instantaneous velocity

$$\bar{\omega} = \frac{\Delta \theta}{\Delta t} = \frac{\theta - \theta_0}{\Delta t}$$

$$a = \frac{dv}{dt} \quad \left(\frac{m}{s^2} \right)$$

instantaneous acceleration

$$\omega = \frac{d\theta}{dt} \quad \left(\frac{\text{rad}}{\text{s}} \right)$$

Equations of motion (constant acceleration)

$$1) v = v_0 + a \cdot t$$

$$4) \omega = \omega_0 + \alpha \cdot t$$

$$2) x = x_0 + v_0 t + \frac{1}{2} a t^2$$

$$2) \theta = \theta_0 + \omega_0 t + \frac{1}{2} \alpha t^2$$

$$3) \frac{v^2 - v_0^2}{x - x_0} = 2a$$

$$3) \frac{\omega^2 - \omega_0^2}{\theta - \theta_0} = 2\alpha$$

2nd Newton's Law

$$\vec{F}_{\text{net}} = \frac{d\vec{p}}{dt}$$

$\vec{p} = m\vec{v}$ linear momentum

$$\vec{\tau}_{\text{net}} = \frac{d\vec{L}}{dt}$$

$\vec{\tau}$: torque (tan) (includes radius in addition to \vec{F})

$\vec{L} = I\vec{\omega}$ angular momentum (includes radius in addition to \vec{p})

I : moment of inertia (includes radius in addition to mass m)

$\vec{\omega}$: angular velocity vector

m is constant

$$\vec{F}_{\text{net}} = m\vec{a}$$

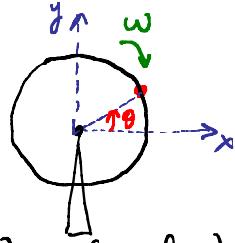
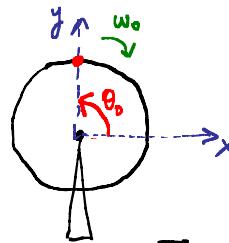


$$\vec{\tau}_{\text{net}} = I\vec{\alpha}$$

$\vec{\alpha}$ = angular acceleration vector

Rotational motion

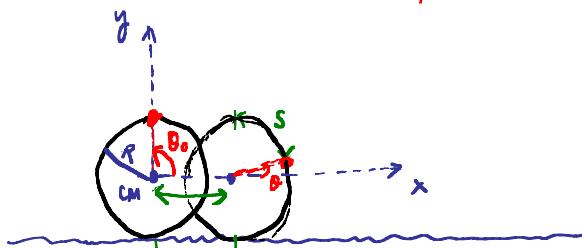
- Change of orientation or angle
- Variables { angle θ (theta)
angular velocity ω (omega)
angular acceleration α (alpha)



Rotational Motion Topics :

- 1) Rolling motion (rotation of object simultaneously with translation of its CM)
- 2). Angular acceleration α
- 3) Torque $\vec{\tau}$ (includes radius in addition to force \vec{F})
- 4) Moment of inertia I (includes radius in addition to mass m)
- 5) KE for rotational motion

1) Rolling motion : There is a quantitative connection b/w translation & rotation
linear vel. & angular vel.



$$\Delta\theta = \theta - \theta_0$$

$$\Delta\theta = \frac{s}{R}$$

- Red dot travelled arc s
- If ball was dipped in ink, it would have traced a distance s on surface
- $\Delta\theta = \frac{s}{R}$; $\left\{ \begin{array}{l} \Delta\theta = \theta - \theta_0 \\ \theta_0: \text{initial angle (1st red dot @ top)} \\ \theta: \text{final angle (2nd red dot)} \end{array} \right.$
 s : distance travelled by CM of ball along x-axis which is a linear motion. → The CM has some linear velocity v
- Note: motion of red dot along perimeter of ball, the arc, dictates the motion of the CM

$$\text{iv) } \frac{d}{dt} [\Delta\theta = \frac{s}{R}]$$

$$\omega = \frac{1}{R} \underbrace{\frac{ds}{dt}}_{\text{linear velocity of CM}} = \frac{1}{R} v$$

Rolling motion

$\boxed{\omega = \frac{v}{R}}$

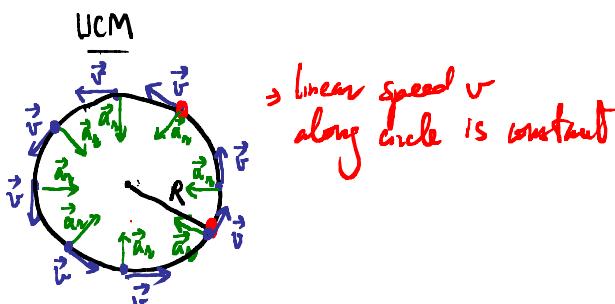
rotation
linear motion

$$\frac{\text{rad}}{\text{s}} = \frac{\text{m}}{\text{s}}$$

or s^{-1}

2) Angular acceleration α

$$\bar{\alpha} = \frac{\Delta\omega}{\Delta t}; \quad \alpha = \frac{d\omega}{dt} \quad \left(\frac{\text{rad}}{\text{s}^2} \right)$$



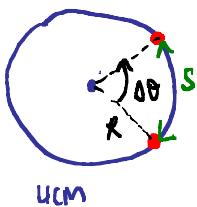
- \vec{v}
- i) same magnitude ~ blue vectors with (Uniform C.M.) same length
 - ii) direction is tangential \rightarrow there is a need of a a_n (radial acceleration toward center of curvature)
- \vec{a} in UCM
- $$\left\{ \begin{array}{l} a_n = \frac{v^2}{R} \quad (v = |\vec{v}| \text{ or linear speed around circle}) \\ a_t = \frac{dv}{dt} = 0 \end{array} \right.$$

What is α in a UCM?

$$\alpha = \frac{d\omega}{dt} = \frac{d(\frac{v}{R})}{dt} = \frac{1}{R} \frac{dv}{dt} = \frac{a_t}{R} = 0$$

$$\boxed{\alpha = \frac{a_t}{R}}$$

$$\omega = \frac{v}{R} : \text{ Proof: } \Delta\theta = \frac{s}{R}$$



$$\frac{d}{dt} [\Delta\theta = \frac{s}{R}]$$

$$\omega = \frac{1}{R} \frac{ds}{dt} = \frac{v}{R}$$

• Similar result to rolling motion but here v

is linear speed along circular trajectory, not of the CM of bowling ball.

non-UCM

→ linear speed v along circle is NOT constant $\rightarrow a_t = \frac{dv}{dt} \neq 0$

$$\vec{a} \cdot \left\{ \begin{array}{l} a_r = \frac{v^2}{R} \quad (\text{now with changing values since it depends on changing } v) \\ a_t = \frac{dv}{dt} \neq 0 \end{array} \right.$$

$$\hookrightarrow \alpha = \frac{a_t}{R} \neq 0$$

$$\leftrightarrow \text{also: } \left[a_t = \frac{dv}{dt} = \frac{d(wR)}{dt} = R \frac{dw}{dt} = R \cdot \alpha \right]$$

3) Torque $\vec{\tau}$ (vector tan)

i) Cross product or \times between two vectors: $\vec{\tau} = \vec{r} \times \vec{F}$

("tan is a cross product of w , r and F ")

ii) Force application point matters \rightarrow need to define clearly $\left\{ \begin{array}{l} \text{pivot or center of rotation} \\ \text{need to know where is the force applied: position of force application point with respect to pivot is described by vector } \vec{r} \end{array} \right.$

iii) Radial vector \vec{r} (from pivot to force application point)

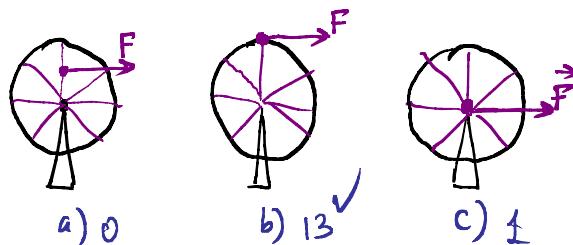
(70)

Linear motion

$$\text{Force } \vec{F} = \text{Force } \vec{F} = \text{Force } \vec{F}$$

Force application point doesn't matter here

Rotation



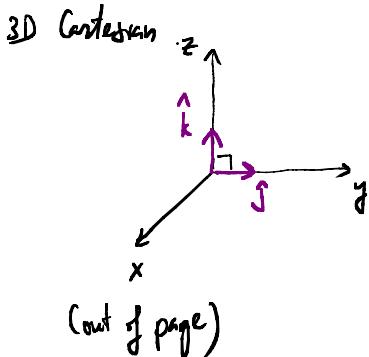
Which force application point gives best speed v
(best \propto to go from $\omega_0=0$ to ω)

iv) Cross-product:

- 1) b/w two vectors (e.g.: $\vec{r} \& \vec{F}$), that produces another vector (e.g.: $\vec{\tau}$)
- 2) direction of the cross-product vector is perpendicular to both vectors ($\vec{\tau} = \vec{r} \times \vec{F}$ is perpendicular to BOTH $\vec{r} & \vec{F}$), + or - as given by the RHR (right-hand rule)
- 3) magnitude of the cross-product is the magnitude of 1st vector times magnitude of 2nd vector times $\sin\theta$ where θ is the angle b/w the two vectors:

$$|\vec{\tau}| = \tau = r F \sin\theta$$

v) RHR: directions of cross-products of Cartesian unit vectors:



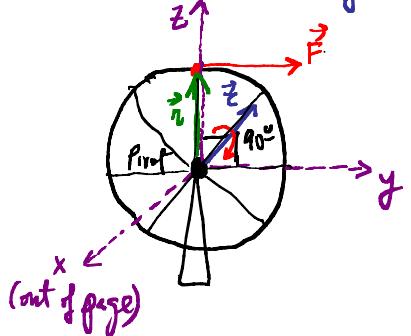
$$a) \hat{k} \times \hat{j} = \begin{cases} \text{magnitude: } 1 \cdot 1 \cdot \sin 90^\circ = 1 \\ \text{direction: by RHR: } \begin{array}{l} 1) \text{ Align RH fingers along 1st vector} \\ 2) \text{ Close RH fingers towards 2nd vector} \\ 3) \text{ Thumb indicates direction of the cross-product} \end{array} \end{cases}$$

$$\rightarrow \hat{k} \times \hat{j} = -\hat{i}$$

$$b) \hat{k} \times (-\hat{j}) = \hat{i} \quad (\hat{j}) \quad \hat{k} \quad \hat{k} \times (\hat{j})$$

$$c) \hat{i} \times \hat{j} = \hat{k} \quad \hat{i} \quad \hat{i} \times \hat{j}$$

RHR to find direction of a torque: $\vec{\tau} = \vec{r} \times \vec{F}$



$$\vec{r} = r\hat{k}$$

$$\vec{F} = F\hat{j}$$

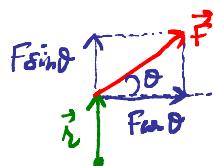
$$\vec{\tau} = rF \underbrace{\hat{k} \times \hat{j}}_{\text{RHR} : \hat{i}} = rF (-\hat{i}) \quad (\text{into page})$$

For a torque that points into page, direction of rotation is also given by a RHR:
if thumbs is along direction of torque, right-hand fingers close in direction of rotation. In this example, a torque along negative x means a CW rotation.

$$\vec{\tau} = \vec{r} \times \vec{F} = rF \underbrace{\hat{k} \times (\hat{j})}_{\text{RHR} : \hat{i}} = rF \hat{i} \quad \rightarrow \text{a torque out of page corresponds to a CCW rotation}$$

$$\vec{\tau} = rF \underbrace{\sin(90-\theta)}_{\cos\theta} \underbrace{(\hat{i})}_{\text{RHR}} = -rF \cos\theta \hat{i} \quad \rightarrow \text{a torque into page gives a CW rotation}$$

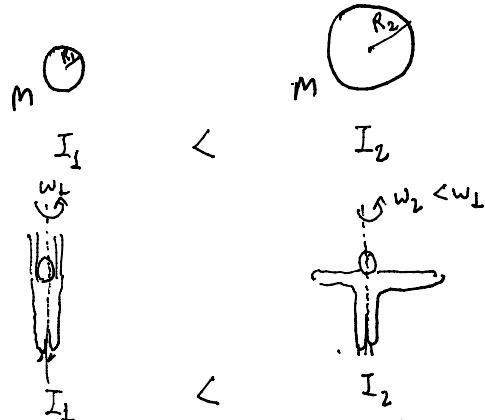
Component of the force applied that is perpendicular to the radial vector \vec{r}



- (i) $F \sin\theta$ is $\parallel \vec{r}$ \rightarrow no torque
- (ii) $F \cos\theta$ is $\perp \vec{r}$ \rightarrow only component that produces torque

If \vec{F} is applied @ angle θ , only part of it ($F \cos\theta$) produces a torque leading to rotational motion.

- 4) Moment of inertia I : (i) Rotational counterpart of mass m
(ii) Includes radius (size does matter)



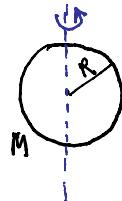
Qualitative formula for I :

$$\left\{ \begin{array}{l} \text{- discrete system : } I = \sum_i m_i r_i^2 \\ \text{- continuous system } I = \int r^2 dm \end{array} \right. \quad \left\{ \begin{array}{l} m_i = \text{mass of component } i \\ r_i = \text{position of } i \text{ wrt.} \\ \text{axis of rotation} \\ dm = \text{infinitesimal mass} \\ r : \text{position of } dm \\ \text{wr. axis of rotation} \end{array} \right.$$

Simple geometrical shapes : sphere, cylinder, ring, disk :

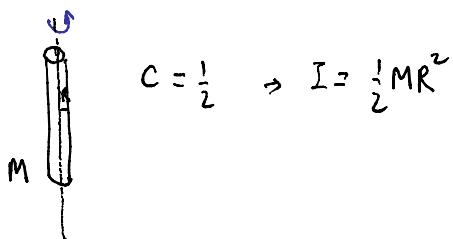
$$I = cMR^2 \quad \left\{ \begin{array}{l} M : \text{total mass of object} \\ R : \text{radius of mass distribution} \\ c : \text{coefficient depending on the actual shape} \end{array} \right.$$

- 1) Sphere wrt its center axis



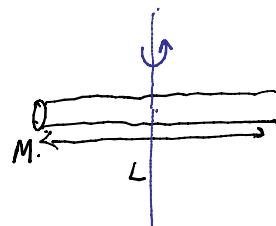
$$c = \frac{2}{5} \rightarrow I = \frac{2}{5} MR^2$$

- 2) Vertical cylinder wrt. its center axis



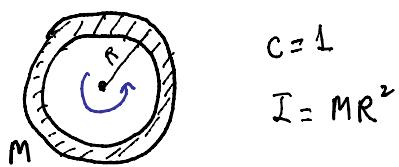
$$c = \frac{1}{2} \rightarrow I = \frac{1}{2} MR^2$$

- 3) Horizontal cylinder wrt. its vertical center axis



$$c = \frac{1}{12} \quad I = \frac{1}{12} ML^2$$

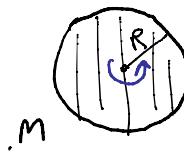
- 4) Ring wrt. its center axis



$$c = 1$$

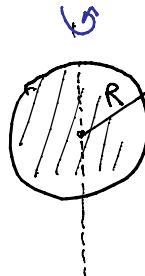
$$I = MR^2$$

5) Uniform disk wrt. its center axis



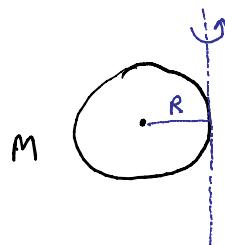
$$c = \frac{1}{2} \rightarrow I = \frac{1}{2} MR^2$$

6) Sphere wrt. its center axis



$$c = \frac{2}{5} \rightarrow I = \frac{2}{5} MR^2$$

Sphere wrt its tangential axis



$$I_{\text{tangential}} = \frac{2}{5} MR^2 + MR^2 = \frac{7}{5} MR^2$$

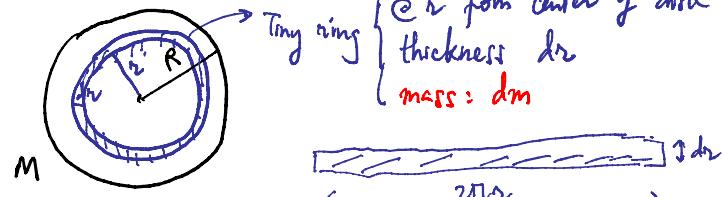
Parallel axis theorem:

$$I_{\text{tangential}} = I_{\text{center axis}} + MR^2$$

R: separation b/w the two axes

Calculation of moment of inertia for uniform disk of mass M, radius R, wrt. its center axis:

$$I = \int r^2 dm$$



Surface areas

(i) tiny ring of radius r & thickness dr is $2\pi r dr$
(ii) disk of radius R is πR^2

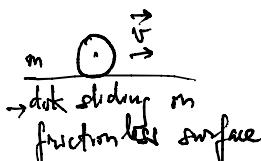
$$\frac{dm}{M} = \frac{2\pi r dr}{\pi R^2}$$

$$I = \frac{2M}{R^2} \int_0^R r^3 dr = \frac{2M}{4R^2} [r^4]_0^R = \frac{2M}{4R^2} R^4 = \frac{1}{2} MR^2 \quad (I = c MR^2)$$

$c = \frac{1}{2}$ for uniform disk wrt. its center axis

5) Kinetic energy in rotational motion:

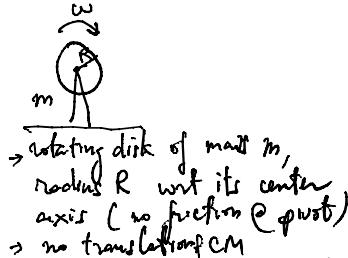
Linear motion



$$KE = \frac{1}{2} mv^2$$

m : inertia for linear motion

Pure rotation



$$KE = \frac{1}{2} I \omega^2, \quad I: \text{inertia in rotations}$$

Rolling motion { Rotation wrt center axis & translation with translation of CM



$$\rightarrow \text{different fraction for rolling motion}$$

$$\rightarrow \left\{ \begin{array}{l} \text{rotation wrt center axis, } I = \frac{1}{2} m R^2 \\ \text{translation of CM } v_{CM} = \omega \cdot R \end{array} \right.$$

$$KE = \frac{1}{2} m v_{CM}^2 + \frac{1}{2} I \omega^2 = \frac{1}{2} m v_{CM}^2 + \frac{1}{2} \left(\frac{1}{2} m R^2 \right) \omega^2$$

$$= \frac{1}{2} m v_{CM}^2 + \frac{1}{2} \left(\frac{1}{2} m \right) v_{CM}^2 = \frac{1}{2} \left(\frac{3}{2} m \right) v_{CM}^2$$

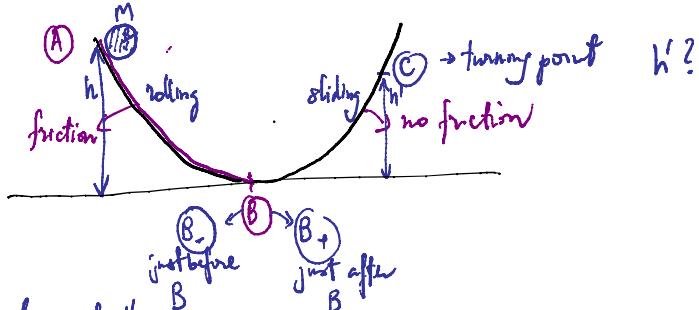
$$\text{Rolling motion: } KE = \frac{1}{2} \left(\frac{2}{5} M \right) v_{cm}^2$$

ABS braking: model each wheel of a car as a disk

\downarrow with rolling motion: each wheel has an effective mass increase of 50% \rightarrow shorter stopping distance

20.64] sphere rolls down left side of a parabolic potential well from height h , then slides up the right side of well where there is no friction, to what height h'

Step 1:



Focus on description of motion of the sphere: (i) left side: friction: rolling sphere only one contact point with surface

friction just enough to allow rotation w/ center axis in addition to translation of CM of sphere

$$(ii) ME_A = M\bar{E}_{B_-} \quad \& \quad M\bar{E}_{B_+} = M\bar{E}_C$$

$$M\bar{E}_{B_+} = M\bar{E}_{B_-} - \frac{1}{2} I \omega^2 \quad (\text{rotational part of KE is on the left side only})$$

$$\downarrow \quad h' < h$$

Step 2:

$$1) \quad \textcircled{A} \quad Mgh = \frac{1}{2} M v_{cm}^2 + \frac{1}{2} I \omega^2$$

$$2) \quad I_{\text{sphere about center axis}} = \frac{2}{5} M R^2$$

$$4) \quad \textcircled{B_+} \quad \frac{1}{2} M v_{cm}^2 = Mgh' \quad (\text{turning point})$$

$$3) \quad \text{Rolling motion b/w } \textcircled{A} \text{ & } \textcircled{B_-}$$

$$v_{cm} = \omega \cdot R \quad \text{or} \quad \omega = \frac{v_{cm}}{R}$$

Step 3:

Solve for h' :

$$4) \quad h' = \frac{v_{cm}^2}{2g}$$

$$v_{cm} \text{ plug } 2) \text{ & } 3) \text{ into } 1) \quad Mgh = \frac{1}{2} M v_{cm}^2 + \frac{1}{2} \left(\frac{2}{5} M R^2 \right) \cdot \left(\frac{v_{cm}}{R} \right)^2$$

$$= \frac{1}{2} \left(M + \frac{2}{5} M \right) v_{cm}^2 = \frac{1}{2} \left(\frac{7}{5} M \right) v_{cm}^2$$

$$Mgh = \frac{1}{2} \left(\frac{7}{5} M \right) v_{cm}^2 \rightarrow \frac{5}{7} h = \frac{v_{cm}^2}{2g}$$

$$4) \quad \boxed{h' = \frac{5}{7} h}$$

Ch 11 Rotational Vectors & Angular Momentum \vec{L}

(75)

Linear Motion

$$2^{\text{nd}} \text{ Newton's law: } \vec{F}_{\text{net}} = \frac{d\vec{p}}{dt}$$

$\vec{p} \equiv m\vec{v}$ linear momentum

$$m = \text{constant: } \vec{F}_{\text{net}} = m \frac{d\vec{v}}{dt} = m\vec{a}$$

$$\begin{array}{l} m \\ \vec{p} \\ \vec{v} \\ \vec{a} \end{array} \longrightarrow \begin{array}{l} I \\ \vec{L} \\ \vec{\omega} \\ \vec{\alpha} \end{array}$$

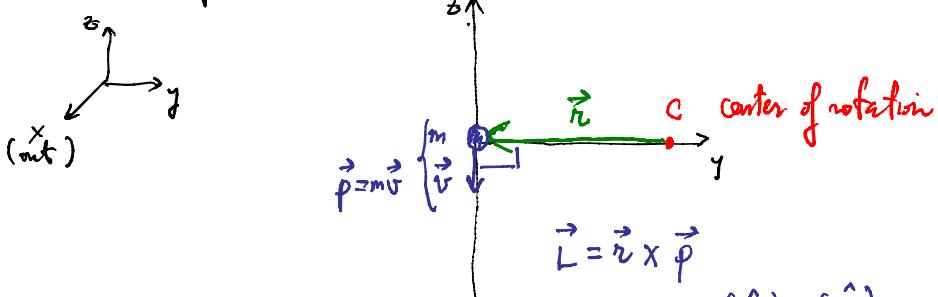
i) So far 2 cross-products in rotational motion $\left\{ \begin{array}{l} \vec{r} = \vec{r} \times \vec{F} \\ \vec{L} = \vec{r} \times \vec{p} \end{array} \right.$

$$\vec{r} = \frac{d\vec{L}}{dt} = \vec{r} \times \frac{d\vec{p}}{dt} = \vec{r} \times \vec{F}$$

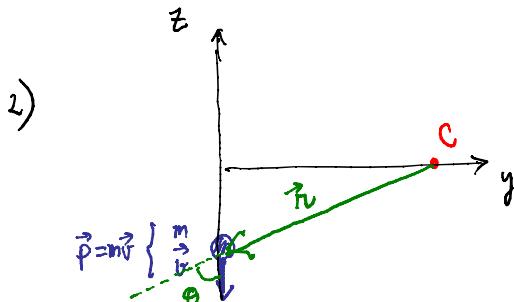
\vec{r} constant

Calculations of angular momentum vector: \rightarrow importance: magnitude & direction

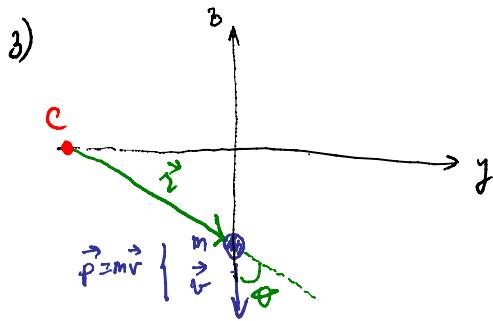
- 1) Object of mass m moving along $-z$ axis, center of rotation @ C



Linear	Rotational
$\vec{F}_{\text{net}} = 0 = \frac{d\vec{p}}{dt}$	$\vec{F}_{\text{net}} = 0 = \frac{d\vec{L}}{dt}$
$\vec{p}_i = \vec{p}_f$	$\vec{L}_i = \vec{L}_f$



The object "rotates" CCW wrt center of rotation C
(smaller than when object was at origin by factor $\sin \theta < 1$)

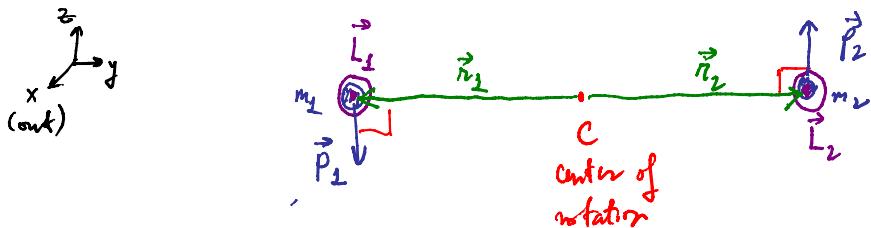


$$\vec{L} = \vec{i} \times \vec{p} = Ipm\theta \hat{i}$$

RHR

Note: \vec{L} depends on selection of center of rotation

- 4) \vec{L}_{Total} for 2 objects of masses m_1 & m_2 moving in opposite directions wrt. middle center of rotation



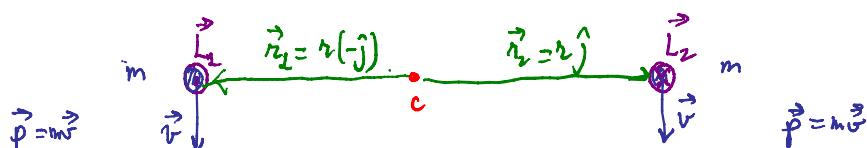
$$\text{Total } \vec{L} = \vec{i} \times \vec{p}_1 + \vec{i} \times \vec{p}_2 = m_1 p_1 \hat{i} + m_2 p_2 \hat{i} = (m_1 p_1 + m_2 p_2) \hat{i}$$

$$\text{If } m_1 = m_2 = m; \quad v_1 = v_2 \equiv v \rightarrow \vec{L}_{\text{Total}} = 2mp \hat{i}$$

$$v_1 = v_2 \equiv v$$

$$\rightarrow p_1 = p_2 \equiv p$$

- 5) Similar to 4) but both masses going in same direction



$$\vec{L}_{\text{Total}} = (m_1 p_1 - m_2 p_2) \hat{i} = 0 \quad \begin{cases} m_1 \text{ rotates about } C \text{ in CCW} \\ m_2 \text{ rotates about } C \text{ in CW} \end{cases}$$

Applications:

Conservation of Angular Momentum:

$$\vec{L}_{\text{init}} = 0 \rightarrow \frac{d\vec{L}}{dt} = 0 \rightarrow \vec{L}_i = \vec{L}_f$$

Initial

Disk w/ mouse
at outer edge
System rotates
wrt. center axis
 $\odot w_L$

Final

Mouse walked to axis of rotation

$$I_{mf} = 0$$

$$L_f = \frac{1}{2} M_d R^2 w_f$$

$$L_f = I_m \cdot w_i + \frac{1}{2} M_d R^2 \cdot w_f$$

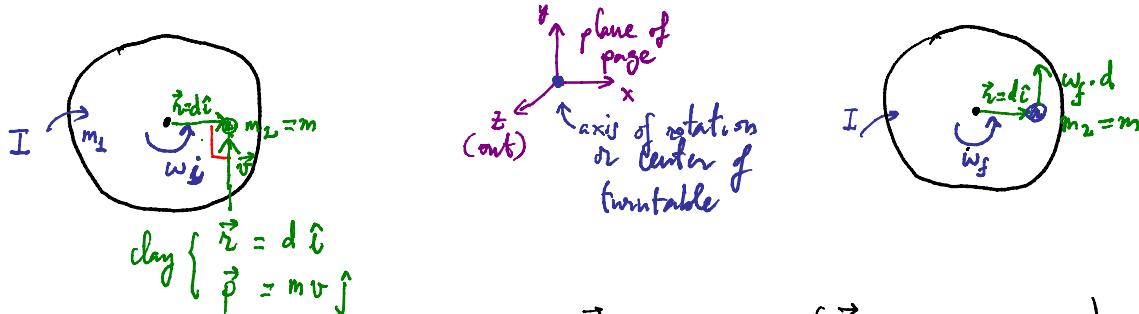
Since there is no net external torque on system of desk & mouse
⑦ $\vec{\tau}_{\text{net}} = 0 \rightarrow \frac{d\vec{L}}{dt} = 0 \rightarrow L_i = L_f$

$$I_m \cdot \omega_i + \frac{1}{2} M_d R^2 \cdot \omega_i = \frac{1}{2} M_d R^2 \omega_f$$

\rightarrow Can solve for $\omega_f > \omega_i$ (less rotational inertia
when mouse stands @ center of rotation)

- 11.53] Turntable, moment of inertia I , rotating @ ω about vertical axis, no friction w/ axis (78)
 Wed of clay of mass m , tossed on, sticks @ d from rotation axis, hits horizontally
 with velocity $v \perp$ radius of turntable, same duration as turntable's rotation.
 $v?$ a) $w_f = \frac{\omega}{2}$
 b) $w_f = \omega$
 c) $w_f = 2\omega$

Step 1: Initial \leftarrow Conservation of angular momentum \rightarrow Final
 clay about to hit turntable View from above clay has just landed of turntable



Step 2: $\vec{L}_i = \vec{L}_f$ ($\vec{\tau}_{\text{net, external}} = 0$)

$$\vec{L}_{2i} + \vec{L}_{ci} = \vec{L}_{2f} + \vec{L}_{cf}$$

$$I\vec{\omega}_i + \vec{r} \times \vec{p} = I\vec{\omega}_f + md^2\vec{\omega}_f$$

$$I\vec{\omega}_i + dm\vec{v} \sin 90^\circ (\hat{i}_x) = (I + md^2)\vec{\omega}_f$$

Note: CCW rotation, by RHR, angular velocity vector points in direction of the thumb when right-hand fingers close in CCW direction. $\rightarrow \vec{\omega}_i = \omega \hat{k}$

$$\rightarrow I\omega + dm\vec{v} = (I + md^2)\vec{\omega}_f \rightarrow v = \frac{(I + md^2)\omega_f - I\omega}{md}$$

Step 3: a) $v?$ $\omega_f = \frac{\omega}{2} \rightarrow v = \frac{[(I + md^2)\frac{\omega}{2} - I]\omega}{md} = \frac{(md^2 - I)\omega}{2md}$

b) $v?$ $\omega_f = \omega \rightarrow v = \frac{(I + md^2 - I)\omega}{md} = d\omega$

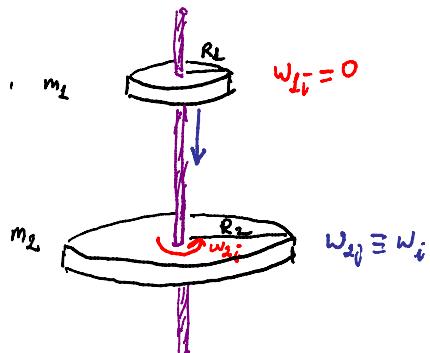
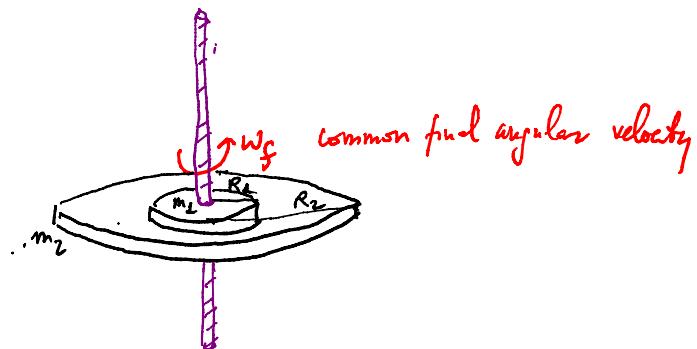
c) $v?$ $\omega_f = 2\omega \rightarrow v = \frac{[(I + md^2)2 - I]\omega}{md} = \frac{I + 2md^2}{md}\omega$

11.57

Two disks $\left\{ \begin{array}{l} m_1 = 0.27 \text{ kg}; R_1 = 0.023 \text{ m}; \omega_{1i} = 0 \\ m_2 = 0.44 \text{ kg}; R_2 = 0.035 \text{ m}; \omega_{2i} = 180 \text{ rpm} \left(\frac{\text{rads}}{\text{min}} \right) \end{array} \right\}$ Top (1) disk is dropped onto rotating bottom (2) disk, giving a common final angular velocity ω_f

Step 1.

$$\vec{L}_{\text{net, external}} = 0 \quad \left\{ \begin{array}{l} \text{System = disk 1 + disk 2} \\ \text{Shaft is external, but frictionless} \\ \Rightarrow \text{Friction b/w ① & ② is internal!} \end{array} \right.$$

InitialFinal

Step 2: Eq. of conservation of angular momentum:

$$(L = Iw) \quad (I_{\text{Disk}} = \frac{1}{2}mR^2)$$

$$\vec{L}_i = \vec{L}_f$$

Same axis of rotation $\Rightarrow L_i = L_f$

$$0 + I_2 w_i = (I_1 + I_2) w_f$$

$$\cancel{\frac{1}{2}m_2 R_2^2} w_i = (\cancel{\frac{1}{2}m_2 R_1^2} + \cancel{\frac{1}{2}m_2 R_2^2}) w_f$$

Step 3: a) solve for w_f :

$$w_f = \underbrace{\frac{m_2 R_2^2}{m_1 R_1^2 + m_2 R_2^2}}_{\ll 1} w_i$$

$$w_f = \frac{0.44 \cdot 0.035^2}{0.27 \cdot 0.023^2 + 0.44 \cdot 0.035^2} 180 \text{ rpm} \approx 142 \text{ rpm}$$

b) Friction b/w ① & ② is what keep them rotating at same w_f : $KE_f = KE_i - KE_{\text{lost}}$

$$\text{lost fraction} = \frac{KE_{\text{lost}}}{KE_i} = \frac{KE_i - KE_f}{KE_i} = 1 - \frac{KE_f}{KE_i} = 1 - \frac{\cancel{\frac{1}{2}I_1 w_f^2} + \cancel{\frac{1}{2}I_2 w_f^2}}{\cancel{\frac{1}{2}I_2 w_i^2}}$$

$$= 1 - \frac{(I_1 + I_2)}{I_2} \left(\frac{w_f}{w_i} \right)^2$$

$$= 1 - \frac{\cancel{\frac{1}{2}m_2 R_2^2} + \cancel{\frac{1}{2}m_2 R_2^2}}{\cancel{\frac{1}{2}m_2 R_2^2}} \left(\frac{w_f}{w_i} \right)^2 = 1 - \frac{0.27 \cdot 0.023^2 + 0.44 \cdot 0.035^2}{0.44 \cdot 0.035^2} \left(\frac{142}{180} \right)^2 = 0.211$$

21.1%

Alternative #2 to calculate the lost fraction: $1 - \frac{KE_f}{KE_i} = 1 - \frac{\frac{1}{2} I_f w_f^2}{\frac{1}{2} I_i w_i^2}$ (80)

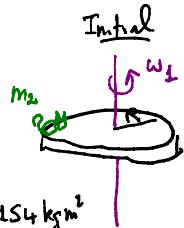
By writing form $\frac{I_i}{I_i w_i} = \frac{L_f}{I_f w_f}$

$$\begin{aligned} \frac{I_i w_i}{\text{Total}} &= \frac{I_f w_f}{\text{Total}} \rightarrow \frac{I_i}{I_f} = \frac{w_f}{w_i} \\ &= I_1 + I_2 \quad \text{or} \quad \frac{I_f}{I_i} = \frac{w_i}{w_f} \rightarrow 1 - \frac{w_i}{w_f} \cdot \frac{w_f}{w_i - w_f} = 1 - \frac{w_f}{w_i} \end{aligned}$$

$$\rightarrow \text{lost fraction} = 1 - \frac{w_f}{w_i} = 1 - \frac{142}{180} = 0.211 \text{ or } 21.1\%$$

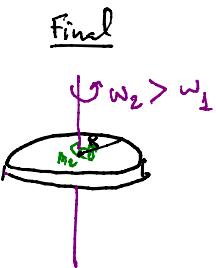
11.42]

Step 1:



$$\sum_{\text{net, ext}} \vec{L} = 0$$

$$\vec{L}_i = \vec{L}_f$$



$$I_L = 0.0154 \text{ kg m}^2$$

$$R = 0.25 \text{ m}$$

$$\omega_1 = 22 \text{ rpm}$$

Rotates freely; no external torque on system

$$m_2 = 19.5 \cdot 10^{-3} \text{ kg}$$

Step 2: a)

$$\vec{L}_{i,i} + \vec{L}_{i,i} = \vec{L}_{i,f} + \vec{L}_{i,f}^\circ$$

$$\vec{L} = \begin{cases} I \vec{\omega} \\ \vec{r} \times \vec{p} \end{cases}$$

$$I_L \omega_1 + m_2 R^2 \cdot \omega_1 = I_L \omega_2 \rightarrow \omega_2 = \frac{I_L \omega_1 + m_2 R^2 \cdot \omega_1}{I_L} = \left(1 + \frac{m_2 R^2}{I_L} \right) \omega_1$$

$$\omega_2 = \underbrace{\left(1 + \frac{19.5 \cdot 10^{-3} \cdot 0.25^2}{0.0153} \right)}_{22 \text{ rpm}} 22 \text{ rpm} = 23.74 \text{ rpm} > \omega_1$$

b) Work done by the mouse: $\Delta KE_f - \Delta KE_i = \frac{1}{2} (I_L + 0) \omega_i^2 - \frac{1}{2} (I_L + m_2 R^2) \omega_f^2$

$$\omega_2 = 23.74 \frac{\text{rad}}{\text{min}} \frac{2\pi \text{ rad}}{1 \text{ rev}} \frac{1 \text{ min}}{60 \text{ s}} = \frac{47.48 \pi}{60} \frac{\text{rad}}{\text{s}}$$

$$\omega_1 = 22 \frac{\text{rad}}{\text{min}} \frac{2\pi \text{ rad}}{1 \text{ rev}} \frac{1 \text{ min}}{60 \text{ s}} = \frac{44 \pi}{60} \frac{\text{rad}}{\text{s}}$$

$$= \frac{1}{2} 0.0154 \cdot \left(\frac{47.48 \pi}{60} \right)^2 - \frac{1}{2} \left(0.0154 + 19.5 \cdot 10^{-3} \cdot 0.25^2 \right) \left(\frac{44 \pi}{60} \right)^2$$

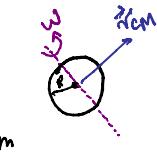
$$= 0.0476 \text{ J} - 0.041 \text{ J} = 0.0036 \text{ J} = 3.5 \text{ mJ.}$$

10.36

Flying baseball $\{m = 0.15 \text{ kg}$
 $R = 3.7 \cdot 10^{-2} \text{ m}$

linear speed $v_{cm} = 33 \frac{\text{m}}{\text{s}}$ angular speed $\omega = 42 \frac{\text{rad}}{\text{s}}$ what fraction of its KE comes from rotational motion (83)

Step 1:



Step 2:

Fraction of rotational KE is

$$\frac{KE_{rotation}}{KE_{cm} + KE_{rotation}} = \frac{\frac{1}{2}I\omega^2}{\frac{1}{2}mv_{cm}^2 + \frac{1}{2}I\omega^2} = \frac{\frac{2}{5}mR^2\omega^2}{\frac{1}{2}v_{cm}^2 + \frac{2}{5}mR^2\omega^2} = \frac{\frac{2}{5}(3.7 \cdot 10^{-2})^2 \cdot 42^2}{33^2 + \frac{2}{5}(3.7 \cdot 10^{-2})^2 \cdot 42^2} = 8.86 \cdot 10^{-4} \approx 0.0886\%$$

very small fraction of total KE was rotational in a flying baseball

Flying baseball

$v_{cm} = 33 \frac{\text{m}}{\text{s}}$

$\omega = 42 \frac{\text{rad}}{\text{s}}$ \rightarrow linear speed of a point on the outer edge of baseball

$v = \omega \cdot R = 42 \cdot 0.0037 = 1.554 \frac{\text{m}}{\text{s}}$
 much smaller compared to v_{cm} !

Rolling baseball

- if same linear speed as flying baseball

$\hookrightarrow \text{Rolling: } v_{cm} = \omega \cdot R \rightarrow \omega = \frac{v_{cm}}{R} = \frac{33}{0.0037} = 892 \frac{\text{rad}}{\text{s}}$

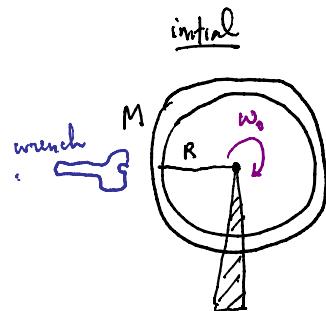
(this is 21.23 times the rotational speed of the flying baseball!)

$$\frac{KE_{rot}}{KE_{cm} + KE_{rot}} = \frac{\omega}{I + \omega} ; I = \omega MR^2 \begin{cases} \text{sphere: } \alpha = \frac{2}{5} \\ \text{disk: } \alpha = \frac{1}{2} \end{cases}$$

$$= \frac{\frac{2}{5}}{1 + \frac{2}{5}} = \frac{2}{7} \approx 0.286 \approx 28.6\%$$

10.58

Step 1:



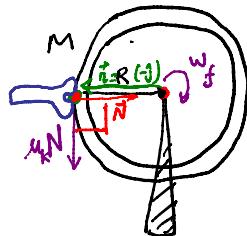
$$w_0 = 230 \text{ rpm}$$

$$M = 1.9 \text{ kg}, \text{ mostly at rim}$$

$$R = 0.33 \text{ m} \quad \text{ring} \rightarrow I = MR^2$$

* System = bike wheel
has a non-zero net torque
applied due to friction b/w
wrench & wheel

final (after wrench is applied
for $\Delta t = 3.1 \text{ s}$)



$$\vec{N} = 2.7 \hat{j} \text{ N} \quad \text{during } \Delta t = 3.1 \text{ s}$$

$$\mu_k = 0.46 \quad (\text{b/w wrench & bike wheel})$$

$$\vec{f}_f = \mu_k \vec{N} (-\hat{k})$$

$\vec{r} = R(-\hat{j})$ position vector of force
application point for \vec{N}
 $\vec{r} = R(-\hat{j})$ position vector of force
application point for \vec{f}_f

$$\begin{aligned} \text{Torque: } \vec{\tau}_{\text{net}} &= \vec{r} \times \vec{N} + \vec{r} \times \vec{f}_f \\ &= RN \sin 180^\circ (-\hat{j}) \times \hat{j} + RM_k N \sin 90^\circ (-\hat{j}) \times (-\hat{k}) \\ \boxed{\vec{\tau}_{\text{net}} = RM_k N \hat{i}} \end{aligned}$$

produces an angular deceleration
Analog of 2nd Newton's Law for rotational motion

Step 2:

$$\tau_{\text{net}} = I\alpha$$

$$RM_k N \hat{i} = MR^2 \frac{w_f - w_0}{\Delta t} \quad (\text{L}) \rightarrow \text{scalar equation: } RM_k N = \frac{MR^2}{\Delta t} (w_0 - w_f)$$

RHR: CW rotation means angular velocities point into the page or $(-\hat{i})$

Step 3:

$$\text{solve for } w_f: \quad w_f = w_0 - \frac{RM_k N \Delta t}{MR^2} = 230 \text{ rpm} - \frac{0.46 \cdot 2.7 \cdot 3.1 \text{ rad}}{1.9 \cdot 0.33} \frac{1 \text{ rev}}{2 \pi \text{ rad}} \frac{60 \text{ s}}{\text{min}} \quad 58.6 \text{ rpm}$$

$$\boxed{w_f = 171 \text{ rpm}}$$

84