

9.28

9.41

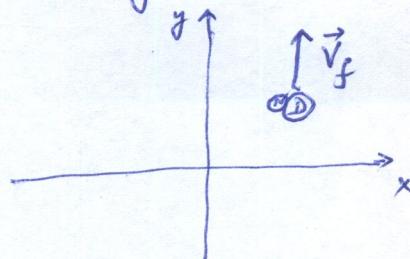
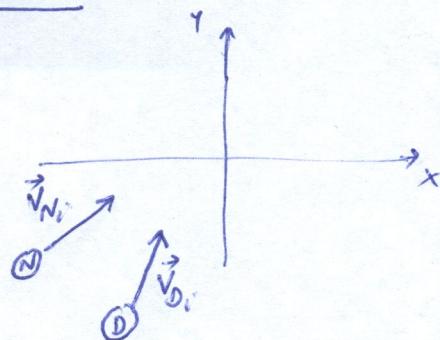
9.45, 9.57

9.63

(9.28)

Neutron ( $1u$ ) striking a Deuteron ( $2u$ ) and combine after collision!

Statement: Inelastic collision : conservation of LM



$$\vec{v}_{N,i} = 28\hat{i} + 17\hat{j} \frac{\text{M}_m}{\text{s}}$$

$$\vec{v}_{D,i} = ?$$

No net external force  $\vec{F}_{\text{net}} = 0 \Rightarrow \vec{P}_i = \vec{P}_f$

$$m \vec{v}_{N,i} + 2m \vec{v}_{D,i} = (m+2m) \vec{v}_f$$

$$\vec{v}_{D,i} = \frac{3 \vec{v}_f - \vec{v}_{N,i}}{2}$$

$$\vec{v}_{D,i} = \frac{36\hat{i} + 60\hat{j} - 28\hat{i} - 17\hat{j}}{2} \frac{\text{M}_m}{\text{s}} = \frac{8\hat{i} - 43\hat{j}}{2} \frac{\text{M}_m}{\text{s}}$$

$$\boxed{\vec{v}_{D,i} = 4\hat{i} - 21.5\hat{j} \frac{\text{M}_m}{\text{s}}}$$

(9.41)

Find the CM of a continuous object  $\leftrightarrow \vec{R} = \frac{1}{m} \int \vec{r} dm$

$\vec{R}$ : position vector of CM

Symmetry: CM on axis of cone

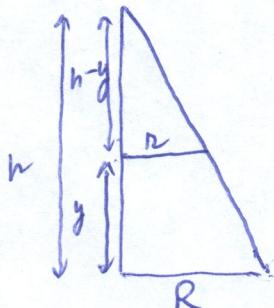


Coord. system centered @  
center of base  $\rightarrow$  only need  
the vertical position for CM!



$$y_{cm} = \frac{1}{m} \int y dm$$

$y$ : vertical position of infinitesimal mass  
 $dm \leftrightarrow$  disk of thickness  $dy$  and  
radius  $r$



Similar triangles:  $\frac{r}{h-y} = \frac{R}{h} \Rightarrow r = R \frac{h-y}{h}$   
 $r = R(1 - \frac{y}{h})$ .

Also:  $dm = \rho dV$   $\left\{ \begin{array}{l} \rho = \text{cone density} \\ dV = \text{element of volume of disk} = \pi r^2 dy \end{array} \right.$

$$\rightarrow dm = \rho \pi r^2 dy$$

$$y_{cm} = \frac{1}{m} \int_0^h y \underbrace{\frac{R^2(1-\frac{y}{h})^2}{r^2}}_{\rho \pi} dy = \frac{\rho \pi R^2}{m} \int_0^h \left(1 - \frac{2}{h}y + \frac{1}{h^2}y^2\right) dy$$

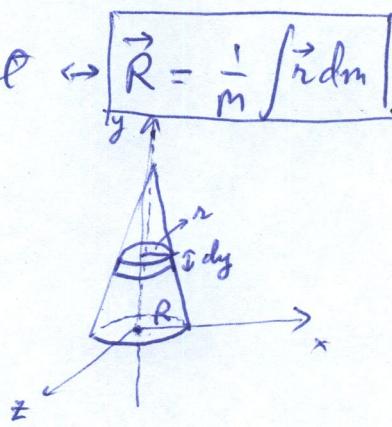
$$= \left(\frac{\rho \pi R^2}{m}\right) \left[ \frac{y^2}{2} - \frac{2}{3h}y^3 + \frac{1}{4h^2}y^4 \right]_0^h = \frac{3}{h} \left[ \frac{h^2}{2} - \frac{2}{3}h^3 + \frac{1}{4}h^4 \right]$$

$$\left. \begin{aligned} \rho_{\text{cone}} &= \frac{M}{\text{Vol}_{\text{cone}}} \\ &= \frac{M}{\frac{\pi R^2 h}{3}} \end{aligned} \right\}$$

$$\frac{\rho \pi R^2}{M} = \frac{M}{\frac{\pi R^2 h}{3}} \cdot \frac{\pi R^2}{M} = \frac{3}{h}$$

$$\begin{aligned} &= 3 \left[ \frac{h}{2} - \frac{2}{3}h + \frac{1}{4}h \right] \\ &= 3h \underbrace{\left[ \frac{1}{2} - \frac{2}{3} + \frac{1}{4} \right]}_{\frac{1}{12}} \end{aligned}$$

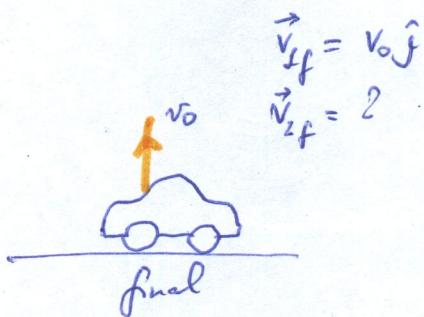
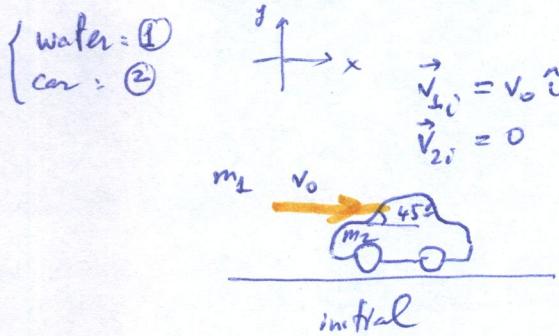
$$y_{cm} = \frac{h}{4}$$



(9.45)

Car initially at rest received a push by a jet of water hitting on its back window. No friction.

(100)



a) Car initial acceleration after receiving hit from water jet?

Car acquired a forward acceleration achieving some final velocity  $\vec{v}_{2f}$

Statements:

- collision b/w jet of water & car  $\rightarrow \vec{P}_i = \vec{P}_f$
- $F_{\text{net}}$  on water & car = 0  $\Rightarrow \vec{P}_i = \vec{P}_f$   
(Friction is ignored)

$$\begin{aligned} \vec{P}_i &= m_1 \vec{v}_{1i} + m_2 \vec{v}_{2i} \\ 3) \quad m_1 \text{ not given but we know the water flow rate is } \frac{dm_1}{dt} & \\ \vec{a}_2 = \frac{d\vec{v}_{2f}}{dt} & \\ \vec{v}_{2f} = \frac{m_1 (\vec{v}_{1i} - \vec{v}_{1f})}{m_2} & \quad \text{constant} \\ \vec{a}_2 = \frac{d\vec{v}_{2f}}{dt} = \frac{1}{m_2} (\vec{v}_{1i} - \vec{v}_{1f}) \frac{dm_1}{dt} & \end{aligned}$$

$$\left\{ \begin{array}{l} \vec{a}_{2x} = \frac{1}{m_2} \frac{dm_1}{dt} v_0 \hat{i} \\ \vec{a}_{2y} = -\frac{1}{m_2} \frac{dm_1}{dt} v_0 \hat{j} \end{array} \right. \quad \begin{array}{l} \text{car} \\ \text{accelerates} \\ \text{forward.} \end{array}$$

(car feels a push downward)

b) Max speed car reaches:

Why can't the jet of water accelerate the car to a very high speed? Relative motion: when car reaches  $v_0$  (same as water speed)  $\rightarrow$  no further push (no more momentum transfer)  $\rightarrow$  max speed for car is  $v_0$ !









Q.57

Tossing a rock standing on ice (No friction)

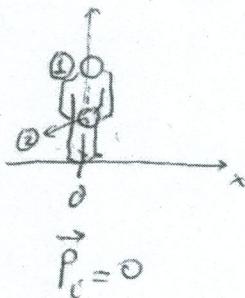
PDI

Conservation law: In.ital

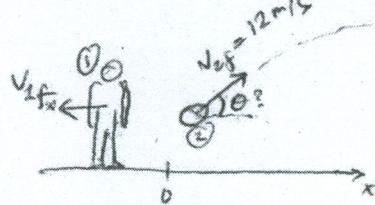
1) System: you + rock

2)  $F_{\text{net}} = 0$ 

$$\rightarrow \vec{P}_i = \vec{P}_f$$

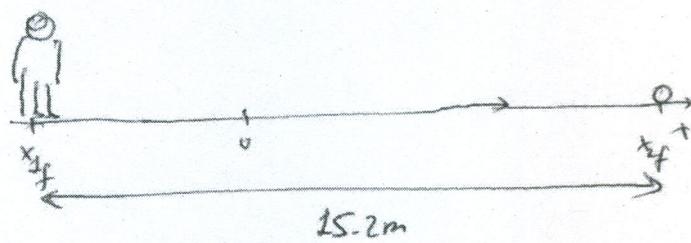
Final

① Uniform motion.

② → Projectile motion.  
 $x = \text{uniform}$   
 $y = \text{constant accel.}$ Find  $\theta$ .Kinematic equation:

$$m_1 = 65 \text{ kg}$$

$$m_2 = 4.5 \text{ kg}$$



$$x\text{-direction: } \left\{ \begin{array}{l} \text{rock: } x_{2f} = v_{2f} \cos \theta \cdot 2t_{\text{up}} \quad (a) \\ x_{2f} - x_{1f} = 15.2 \text{ m} \end{array} \right.$$

$$\left. \begin{array}{l} \text{you: } x_{1f} = v_{1fx} \cdot 2t_{\text{up}}. \quad (b) \end{array} \right.$$

$t_{\text{up}}$ : time for rock to go to max altitude point : final speed in  $y$ -direction of 0 :

$$v_{2fy} = v_{2f} \sin \theta - g \cdot t$$

$$\downarrow 0 = v_{2f} \sin \theta - g \cdot t_{\text{up}} \rightarrow t_{\text{up}} = \frac{v_{2f} \sin \theta}{g}$$

$$\rightarrow \text{Plug back into (a)} \quad x_{2f} = v_{2f} \cos \theta \cdot 2 \cdot \frac{v_{2f} \sin \theta}{g}$$

$$(x_{\text{range}} = \frac{v_0^2 \sin 2\theta}{g})$$

$$= \frac{v_{2f}^2 2 \cos \theta \sin \theta}{g} = \frac{v_{2f}^2 \sin(2\theta)}{g}$$

$$x_{2f} = \frac{12 \sin(2\theta)}{9.81} \quad (x_{2f} \text{ in term of } \theta)$$

(b) Writing  $x_{1f}$  in term of  $\theta$ :

rock

person & rock are related  
in conservation of Total  
momentum!  $\vec{P}_i = \vec{P}_f$

$$\vec{P}_i = \vec{P}_f$$

$$0 = m_1 \vec{v}_{if} + m_2 \vec{v}_{if} \quad \left\{ \begin{array}{l} \text{Rock follows 2D motion} \\ \text{Person acquired } \vec{v}_{if} \text{ but} \\ \text{only } v_{ifx} \text{ showed} \\ \text{just adds pressure} \\ \text{in his feet!} \end{array} \right.$$

X-direction:  $0 = m_1 v_{ifx} + m_2 v_{ifx}$   
 Y-direction:  $0 = m_1 v_{ify} + m_2 v_{ify}$

$$\rightarrow 0 = m_1 v_{ifx} + m_2 v_{if} \cos\theta \rightarrow v_{ifx} = - \frac{m_2}{m_1} v_{if} \cos\theta$$

$$x_{if} = v_{ifx} \cdot 2t_{up} = - \frac{m_2}{m_1} v_{if} \cos\theta \cdot 2 \frac{v_{if} \sin\theta}{g}$$

$$x_{if} = - \frac{m_2}{m_1} \frac{v_{if}^2 \sin(2\theta)}{g}$$

$$x_{if} - x_{if} = 15.2 \text{ m}$$

$$\frac{12^2 \sin(2\theta)}{9.81} + \frac{4.5}{65} \frac{12^2 \sin(2\theta)}{9.81} = 15.2 \text{ m}$$

$$\sin(2\theta) = \frac{15.2}{\left[ \frac{144}{9.81} \left( 1 + \frac{4.5}{65} \right) \right]}$$

$$\theta = \frac{1}{2} \sin^{-1} [ 0.974 ]$$

$$\theta = 38.5^\circ$$

9.63

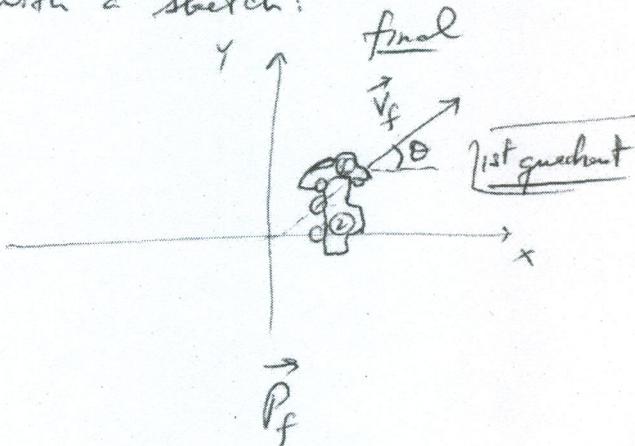
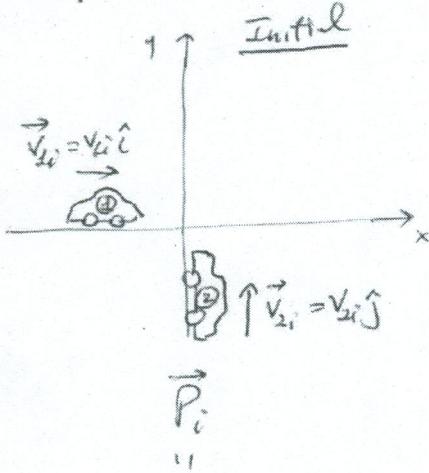
a) System:

$m_1 = 1200\text{kg}$   $m_2 = 2200\text{kg}$   
 Toyata + Buck; inelastic then  
lock together & skid 22m  
colliding @ right angle.  $\rightarrow$  2D

$$\mu_K = 0.91$$

Show at least one car exceeded  $25 \frac{\text{km}}{\text{h}}$  speed limit.

CLM  $\vec{P}_i = \vec{P}_f \rightarrow$  b) Define initial & final situations with a sketch:

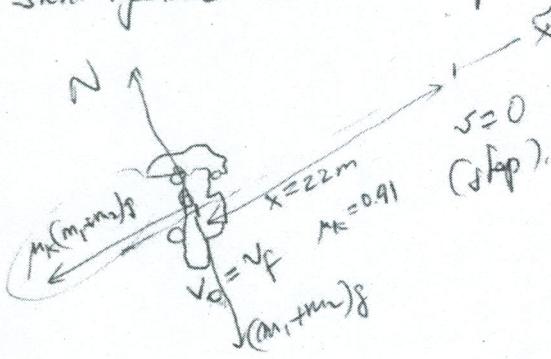


$$I) m_1 v_{1i} \hat{i} + m_2 v_{2i} \hat{j} = (m_1 + m_2) (v_f \cos \theta \hat{i} + v_f \sin \theta \hat{j})$$

$$(I) m_1 v_{1i} = (m_1 + m_2) v_f \cos \theta \rightarrow 1200 v_{1i} = 3400 v_f \cos \theta$$

$$(II) m_2 v_{2i} = (m_1 + m_2) v_f \sin \theta \rightarrow 2200 v_{2i} = 3400 v_f \sin \theta$$

2) Skid together 22m to stop ( $b/c$  of friction  $\mu_K = 0.91$ )



They will come to a stop when all kinetic energy has been used to overcome friction:

$$\frac{1}{2} (m_1 + m_2) v_f^2 = \mu_K (m_1 + m_2) g \cdot x$$

$$v_f = \sqrt{2 \cdot \mu_K g x} = \\ = \sqrt{2 \cdot 0.91 \cdot 9.81 \cdot 22} =$$

$$\boxed{v_f = 19.82 \frac{\text{m}}{\text{s}}}$$

$$(I) \quad 1200 v_{1i} = 3400 v_f \cos \theta \rightarrow \cancel{\text{cancel}}$$

$$(II) \quad 2200 v_{2i} = 3400 v_f \sin \theta$$

$$1200^2 v_{1i}^2 + 2200^2 v_{2i}^2 = (3400 \cdot 19.82)^2 \left( \underbrace{\cos^2 \theta + \sin^2 \theta}_1 \right)$$

$$v_{1i}^2 + \underbrace{\left( \frac{2200}{1200} \right)^2 v_{2i}^2}_{3.36} = \underbrace{\left( \frac{3400 \times 19.82}{1200} \right)^2}_{3150}$$

$$v_{1i}^2 + 3.36 v_{2i}^2 = 3150$$

$$\text{Now: speed limit was } 25 \frac{\text{km}}{\text{h}} = \frac{25}{3.6} \frac{\text{m}}{\text{s}} = 6.94 \frac{\text{m}}{\text{s}}$$

Hypothesis: Each car was traveling @  $6.94 \frac{\text{m}}{\text{s}} \approx 7 \frac{\text{m}}{\text{s}}$

$49 + 3.36 \cdot 49 \approx 200 \rightarrow$  clearly at least one car was traveling well above  $25 \frac{\text{km}}{\text{h}}$

9.74

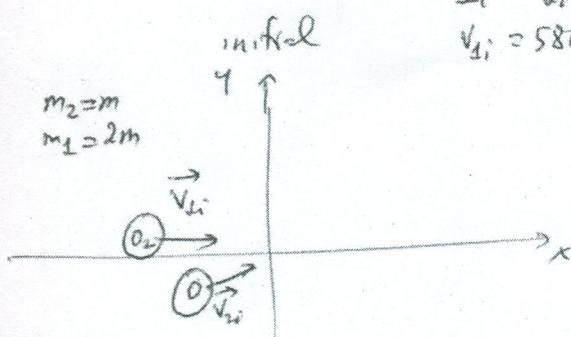
System:

$$\text{O}_2 \text{ & } O \\ (m_1 = 32\text{u}) \quad (m_2 = 16)$$

inelastic collision.

stick together  $\rightarrow$  Ozone

105

 $v_f$ ?

$$\vec{F}_{\text{net}} = 0 \rightarrow \vec{P}_i = \vec{P}_f$$

$$2m \cdot 580 \hat{i} + m \cdot 870 (\cos 27^\circ \hat{i} + \sin 27^\circ \hat{j}) = 3m (v_{fx} \hat{i} + v_{fy} \hat{j})$$

$$1160 + 870 \cos 27^\circ = 3 v_{fx}$$

$$870 \sin 27^\circ = 3 v_{fy}$$

$$v_{fx} = \frac{1160 + 870 \cos 27^\circ}{3} = 645.06 \frac{\text{m}}{\text{s}}$$

$$v_{fy} = \frac{870 \sin 27^\circ}{3} = 131.66 \frac{\text{m}}{\text{s}}$$

Cartesian  $\rightarrow$  polar:

$v_f = \sqrt{645.06^2 + 131.66^2} = 658.4 \frac{\text{m}}{\text{s}}$
(ozone)
$\theta = \tan^{-1} \frac{131.66}{645.06} = 11.54^\circ$

9.51

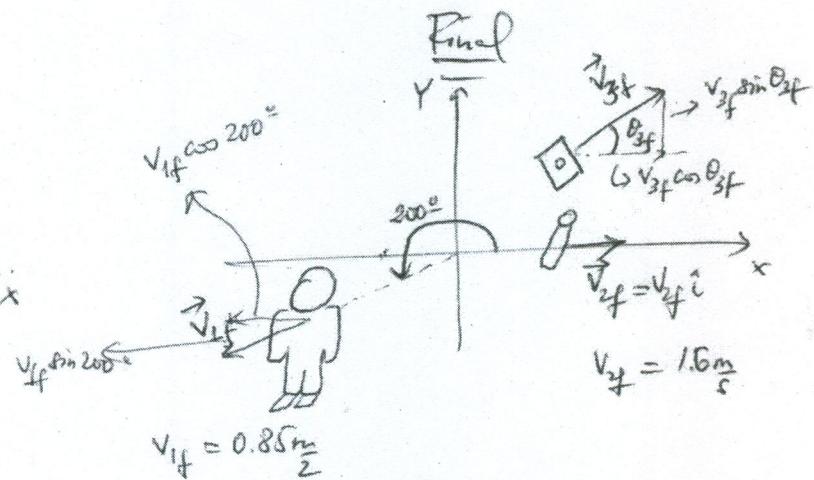
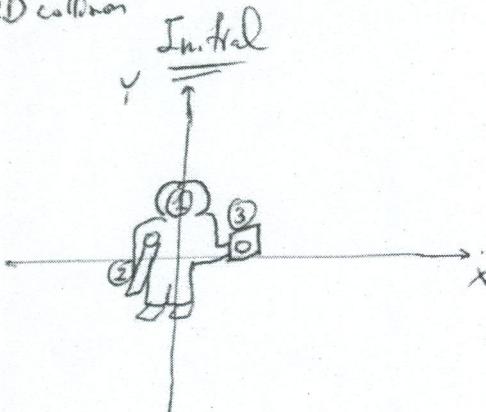
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3 component system : astronaut + O<sub>2</sub> tank + camera

recoils  
at 200° ccw  
from x-axis  
→ 2D collision

In space

$$\rightarrow \vec{F}_{\text{net, external}} = 0 \rightarrow \vec{P}_i = \vec{P}_f$$



$$v_{1f} = 0.85 \frac{m}{s}$$

$$\vec{P}_i = 0 = \vec{P}_f = m_1 \vec{v}_{1f} + m_2 \vec{v}_{2f} + m_3 \vec{v}_{3f}$$

$$0 = (m_1 v_{1f} \cos 200^\circ + m_2 v_{2f} + m_3 v_{3f} \cos \theta_3f) \hat{i} + (m_1 v_{1f} \sin 200^\circ + m_3 v_{3f} \sin \theta_3f) \hat{j}$$

$$0\hat{i} + 0\hat{j}$$

$$\begin{cases} x\text{-direction} : 0 = 60 \cdot 0.85 \cos 200^\circ + 14 \cdot 1.6 + 5.8 v_{3f} \cos \theta_3f \\ y\text{-direction} : 0 = 60 \cdot 0.85 \sin 200^\circ + 5.8 v_{3f} \sin \theta_3f \end{cases}$$

$$\equiv v_{3fx}$$

$$\equiv v_{3fy}$$

$$v_{3fx} = \frac{-60 \cdot 0.85 \cos 200^\circ - 14 \cdot 1.6}{5.8} = 4.4 \frac{m}{s}$$

$$v_{3fy} = \frac{-60 \cdot 0.85 \sin 200^\circ}{5.8} = 3 \frac{m}{s}$$

1st quadrant.

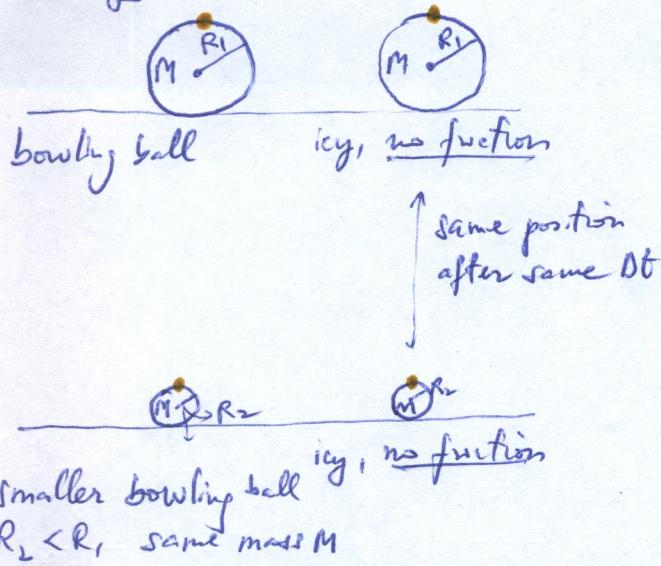
$$\text{Cartesian} \rightarrow \text{Polar} \rightarrow \left\{ \begin{array}{l} v_{3f} = \sqrt{4.4^2 + 3^2} = 5.33 \frac{m}{s} \\ \theta_{3f} = \tan^{-1} \frac{3}{4.4} = 34.3^\circ \end{array} \right. \quad \begin{array}{l} (\text{ccw from } +x\text{-axis}) \\ \text{1st quadrant} \end{array}$$

## Ch.10 Rotational Motion

So far: linear motion, circular motion (object going around an external center of curvature)

### Translation (linear motion)

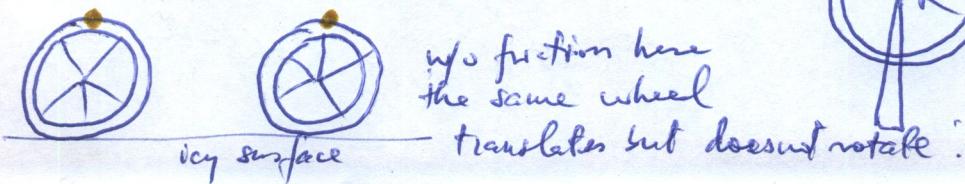
sliding



Statements: 1) sliding balls of equal masses but different radii  $R_2 < R_1$ , have same translational motion

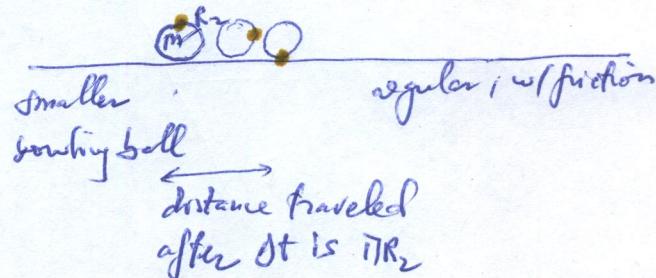
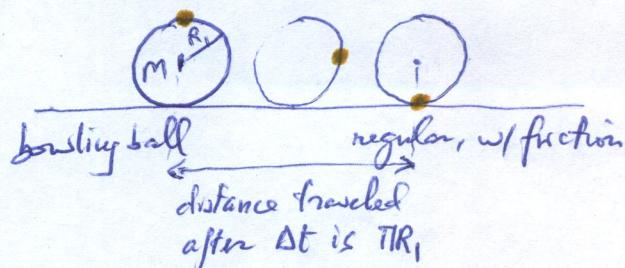
since they can be described as pointlike particles located at their centers of mass with mass equals to  $M$  (radius does not matter)

2) translational motion: same orientation: top dot always stay the same



### Rotation

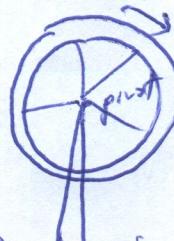
Non-sliding: Rolling



Statements = 1) In rotational motion radius does matter!  
2) Rolling motion involves both translation (note the distance traveled) & rotation

↓  
Is there any pure rotational motion? Is it always connected with a translation?

No! → flipped tire wheel: on a support rotates but does not translate.



- 1) Car wheels under normal road conditions: rolling motion  
 $\Leftrightarrow$  both translation & rotation
- 2) Car wheels stuck in sand, trying to get out: only rotation
- 3) ABS braking : anti-blocking system: when brake pedal is pressed, wheels are not instantly blocked (as in bike braking)  $\Rightarrow$  although we have to increase the pressure to avoid a dry stop @ high speed  
 Wheels makes a few rotations before stopping
  - (i) Car wheels w/ brake applied w/o ABS : only translation
  - (ii) Car wheels w/ brake applied w/ ABS : both translation & rotation

To come to a <sup>full</sup> stop we need to lose all KE:

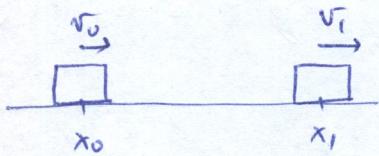
  - (i) Relying on friction only
  - (ii) In addition to friction some of the KE goes into the final few rotation of the wheels  $\rightarrow$  shorter stopping distance (a little bit but can save lives)

$\rightarrow$  better vehicle control when braking at curves.

Guide on previous knowledge: translation

### Translational Motion

→ change of position



$$\bar{v} = \frac{v_0 + v_1}{2}$$

$$\bar{v} = \frac{\Delta x}{\Delta t} \quad (\text{average velocity})$$

$$v = \frac{dx}{dt} \quad (\text{instantaneous velocity})$$

$$a = \frac{dv}{dt}$$

### Equations of motion: (Ch 2 & 3)

$$1) \quad v = v_0 + a \cdot t$$

$$2) \quad x = x_0 + v_0 \cdot t + \frac{1}{2} a t^2$$

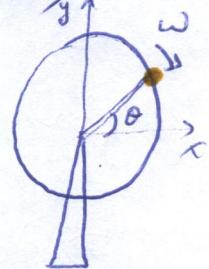
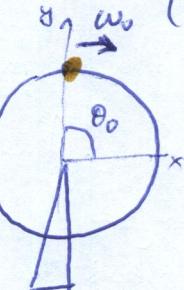
$$3) \quad \frac{v^2 - v_0^2}{x - x_0} = 2a$$

2nd Newton's law (Ch 4)

$$F_{\text{net}} = m \cdot a \\ (\text{constant } m)$$

### Rotational motion

→ change of angle  
(or orientation)



$\omega_0$ : initial angular speed  
(omega sub zero)

$\omega$ : final angular speed.  
(omega)

$$\bar{\omega} = \frac{\omega_0 + \omega}{2}$$

$$\bar{\omega} = \frac{\Delta \theta}{\Delta t} \quad (\text{average angular velocity})$$

$$\omega = \frac{d\theta}{dt} \quad (\text{instantaneous})$$

SI units:  $\frac{\text{radians}}{\text{s}}$  or  $\frac{\text{rad}}{\text{s}}$

$$\alpha = \frac{d\omega}{dt} \quad \left( \frac{180^\circ}{\pi} \right) \quad (\text{angular acceleration})$$

SI units:  $\frac{\text{rad}}{\text{s}^2}$

$$1) \quad \omega = \omega_0 + \alpha \cdot t$$

$$2) \quad \theta = \theta_0 + \omega_0 \cdot t + \frac{1}{2} \alpha t^2$$

$$3) \quad \frac{\omega^2 - \omega_0^2}{\theta - \theta_0} = 2\alpha$$

Analog of 2nd Newton's law

$$F_{\text{net}} = I \cdot \alpha$$

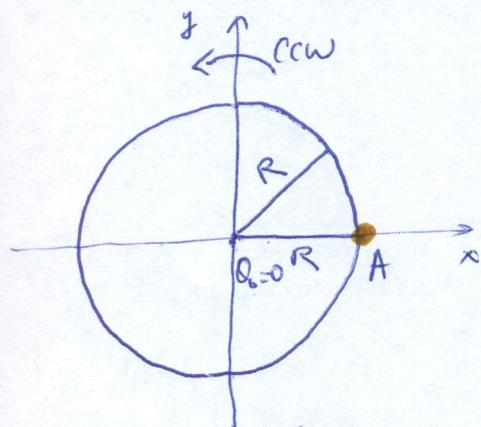
$F_{\text{net}}$ : net torque (radius does matter);  $I$ : moment of inertia (radius does matter)

New items: torque (related to force but radius does matter)  
 moment of inertia (related to mass but radius does matter)

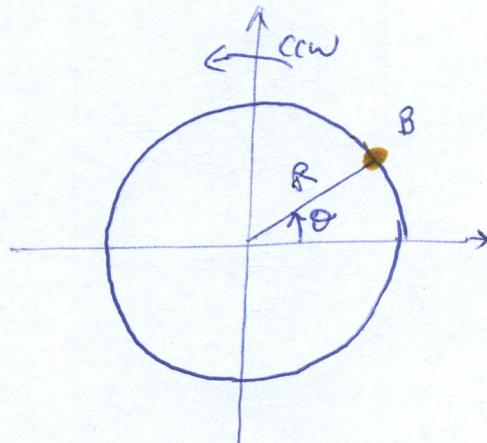
1) Rolling motion: quantitative connection b/w translation & rotation  
 (linear vel.) (angular vel.)



Focus on a point on bowling ball & how its motion is related to the translation of center of bowling ball



center of ball is @  
 origin of coord.



a) In st : arc AB is  $s$  (whch angle went from 0 to  $\theta$ )

$$\text{rotation} \leftarrow \theta = \frac{s \leftarrow \text{translation}}{R}$$

In rolling motion: displacement of center of mass equals the ~~other~~ arcs

b) Connection b/w  $\omega$  &  $v$ :

$$\frac{d}{dt} \left[ \theta = \frac{s}{R} \right]$$

$$\boxed{\omega = \frac{v}{R}}$$



$$\left( \frac{ds}{dt} = v \right)$$

$R$  is constant

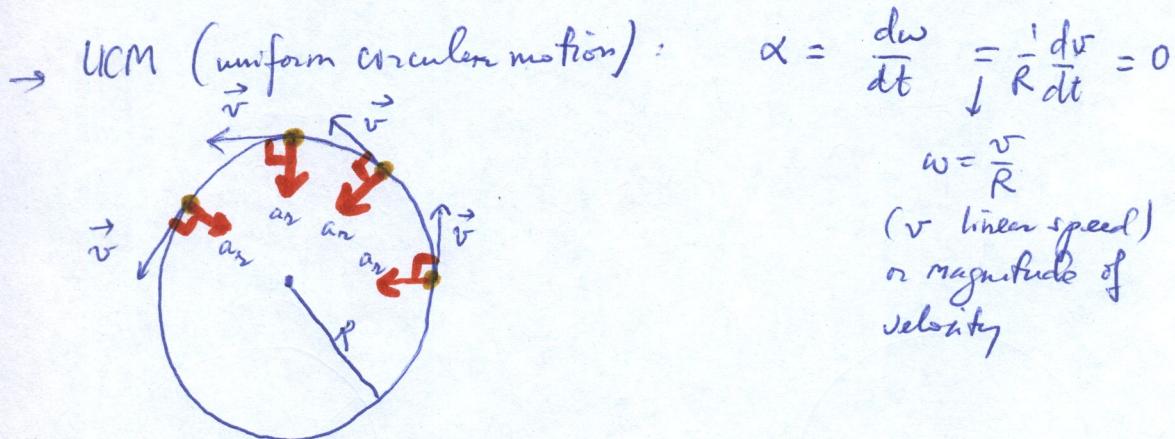
$$\frac{\text{rad}}{s}$$

$$\frac{\frac{m}{s}}{m} = \frac{1}{s}$$

angle is considered dimensionless  $[\theta] = 1$

## 2) Angular acceleration $\alpha$ (alpha)

$$\bar{\alpha} = \frac{d\omega}{dt} ; \quad \alpha = \frac{d\omega}{dt} \quad \left( \frac{\text{rad}}{\text{s}^2} = \frac{1}{\text{s}^2} = \text{s}^{-2} \right)$$



$$\omega = \frac{v}{R}$$

( $v$  linear speed)  
or magnitude of velocity

$\rightarrow$  Non-UCM (non-uniform circular motion): linear speed  $v$  along circle is not constant  $\rightarrow \alpha = \frac{d\omega}{dt} = \frac{1}{R} \frac{dv}{dt}$

UCM {

- $a_r = \frac{v^2}{R}$  (radial acceleration)
- $a_t = \frac{dv}{dt} = 0$  (tangential acceleration)
- $\alpha = 0$

$\rightarrow$  Non-UCM (non-uniform circular motion): linear speed  $v$  along circle is not constant  $\rightarrow \alpha = \frac{d\omega}{dt} = \frac{1}{R} \frac{dv}{dt}$

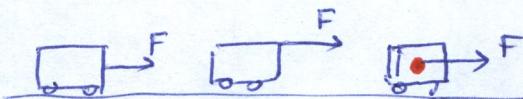
Non-UCM {

- $a_r = \frac{v^2}{R}$  (radial acceleration)
- $a_t = \frac{dv}{dt}$  (tangential acceleration)  $= \frac{d(\omega R)}{dt} = R \frac{d\omega}{dt} = R\alpha$
- $\alpha \neq 0$

$$\omega = \frac{v}{R} \Rightarrow [v = \omega R]$$

3) Torque:  $\vec{\tau} (\text{Nm})$  (radius matters)

In linear motion:



(object can be described as a point @ its center of mass)

In rotational motion:



Good rotation:

Not c)  $\vec{F}$  along radial direction does not help rotation (direction of  $\vec{F}$  is important as it is vector)

b) Not just direction of  $\vec{F}$  but where it is applied is important for rotation! (radius does matter!)

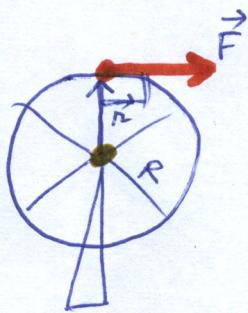
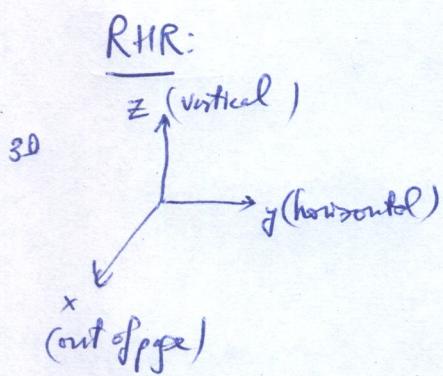
$$\vec{\tau} = \vec{F} \times \vec{r}$$

$\vec{\tau}$ : "cross product" (different than the scalar product in work)  
 a product b/w 2 vectors that produces another vector that is perpendicular to both  
 $\vec{r}$  = position vector of the force application point.  
 wrt. pivot or center of rotation  
 ( $\vec{r}$  goes from pivot to the force application point)

$= r F \sin\theta \hat{\vec{\tau}}$   
 $\theta$ : angle b/w  $\vec{r}$  &  $\vec{F}$   
 $\hat{\vec{\tau}}$ : unit vector in the direction of  $\vec{\tau}$ :  
 perpendicular to plane formed by  $\vec{r}$  &  $\vec{F}$   
 direction given by the right hand rule (RHR)

Unit: Nm (same as work)

Torque is a vector whose magnitude is the product of the magnitudes of the position vector  $\vec{r}$  and the force applied  $\vec{F}$  times the sine of the angle  $\theta$  b/w those two vectors. Direction of torque is perpendicular to both  $\vec{r}$  &  $\vec{F}$  and given by RHR



$$\vec{n} = \hat{R}\hat{k}$$

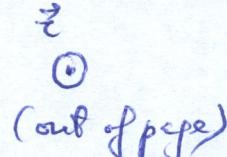
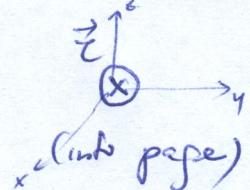
$$\vec{F} = F\hat{j}$$

What is  $\vec{\tau}$ ?  $\vec{\tau} = RF \underbrace{\sin 90}_{1} \hat{k} \times \hat{j} = RF \hat{k} \times \hat{j}$

RHR on  $\hat{k} \times \hat{j}$ : align RH fingers along 1<sup>st</sup> vector  $\hat{k}$  as you close your RH fingers toward the 2<sup>nd</sup> vector  $\hat{j}$  or  $\vec{F}$ , RH thumb points in direction of  $\hat{k} \times \hat{j}$  or  $\vec{\tau}$

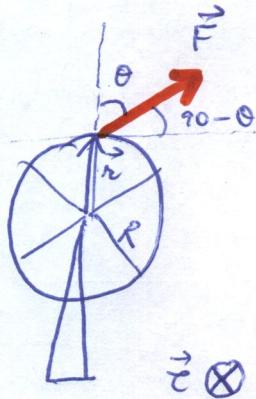
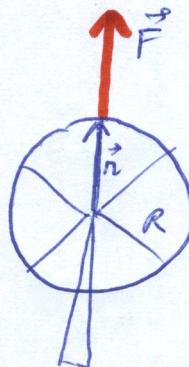
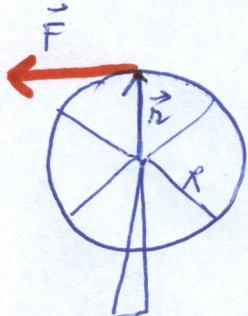
In this case: thumb points into the page or  $(-\hat{i})$   
this is the direction of  $\hat{k} \times \hat{j}$  or  $\vec{\tau}$

Convention:



$$\vec{\tau} = -RF \hat{i}$$

↓  
this case



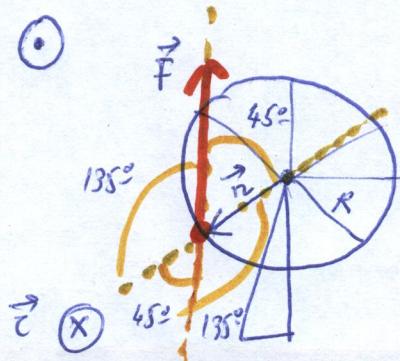
$$\vec{\tau} = RF \hat{i}$$

$$\vec{\tau} = RF \underbrace{\sin 0}_{0} (\hat{k} \times \hat{k}) = 0$$

$$\vec{\tau} = RF \sin \theta (-\hat{i})$$

$$= -RF \underbrace{\sin \theta}_{< RF} \hat{i}$$

Angle  $\theta$ :



Force is applied @ mid point of 3rd quadrant.

$$\vec{\tau} = RF \sin \theta (-\hat{i}) = RF |\sin \theta| (-\hat{i})$$

magnitude is always positive

$\sin 45^\circ = -\sin 135^\circ \Rightarrow$  take absolute value in  $\sin \theta$ .

1) Rolling motion 2) Angular acceleration  $\alpha$  3) Torque  $\vec{\tau}$  4) Moment of inertia  $I$

$\downarrow$   
 $\vec{F} \& \vec{\tau}$

$m \& r$

radius does matter in  
rotational motion.

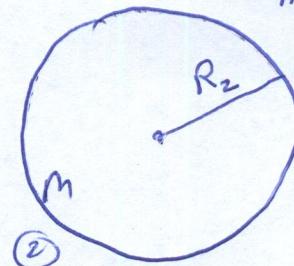
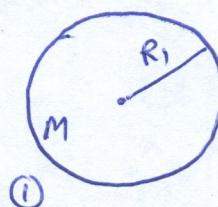
$$\vec{\tau}_{\text{net}} = I\alpha \quad (\vec{F}_{\text{net}} = ma)$$

inertia for  
rotation  
motion.

inertia for  
linear motion  $m$

#### 4) Moment of Inertia I (

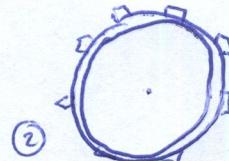
→ Size:



spheres with equal masses but different distributions or density  $R_1 < R_2$

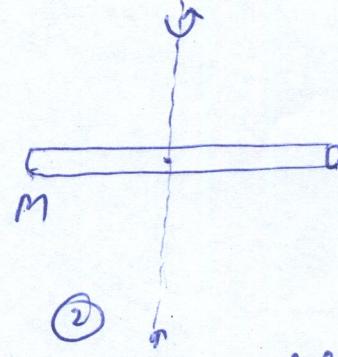
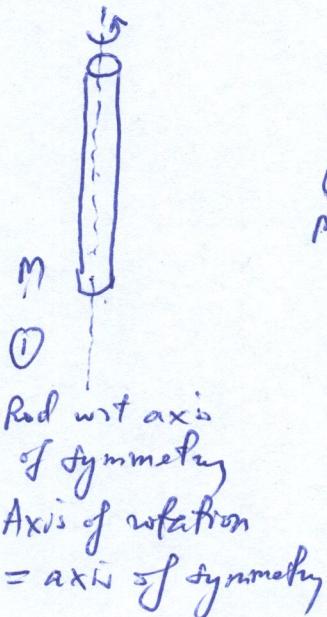
② offers more inertia  
to rotation  
although both would  
have same inertia to  
linear motion.

bike gears



② is harder to  
pedal (more I)

→ Axix of rotation:



② offers more I

$$I = \begin{cases} \text{Discrete systems: } & I = \sum_i m_i r_i^2 \\ \text{Continuous systems: } & I = \int r^2 dm \end{cases}$$

Simple geometrical objects: disk, rods, spheres, ...

$$\hookrightarrow I = \alpha MR^2 \quad \left\{ \begin{array}{l} M: \text{total mass} \\ R: \text{radius of the mass distribution} \\ \text{wrt center / axis of rotation} \end{array} \right.$$

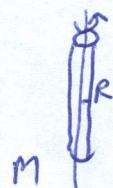
↓  
constant

- 1) Sphere wrt center axis



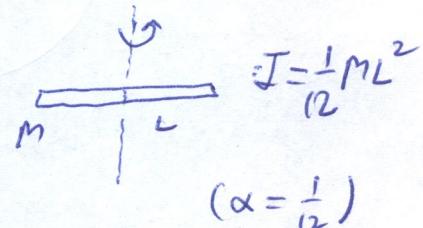
$$I = \frac{2}{5} MR^2 \quad (\alpha = \frac{2}{5})$$

- 2) Cylinder " " "



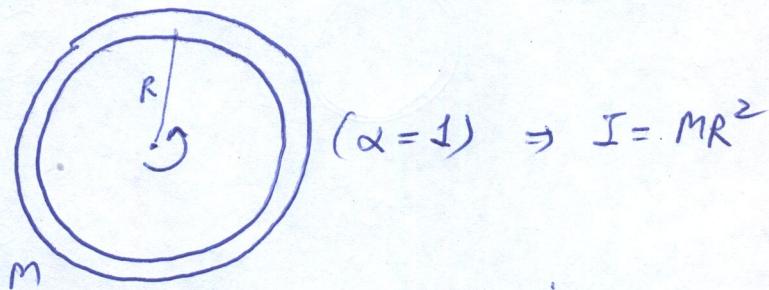
$$I = \frac{1}{2} MR^2 \quad (\alpha = \frac{1}{2})$$

- 3) Thin rod of length  $L$  wrt middle axis



$$I = \frac{1}{12} ML^2$$

- 4) Ring of mass  $M$  & radius  $R$  (bike wheel)



$$(\alpha = 1) \Rightarrow I = MR^2$$

- 5) Sphere wrt a tangential axis:



Parallel axis theorem:

$$I = I_{\text{center axis}} + MR^2$$

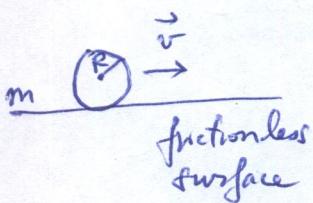
( $R$  is sep. b/w  
the tangential axis  
& the center axis)

$$= \frac{2}{5} MR^2 + MR^2$$

$$I = \frac{7}{5} MR^2$$

## Kinetic Energy in rotational motion:

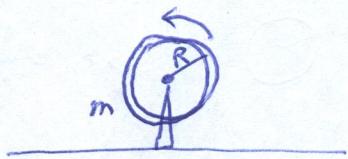
①

linear motion

Disk sliding on  
frictionless surface  
with velocity  $\vec{v}$

$$KE = \frac{1}{2}mv^2$$

②

Pure rotational motion

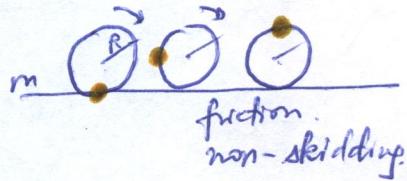
Disk rotating freely  
on a support  
(no translation)

$$KE = \frac{1}{2}I\omega^2$$

③

Rolling motion

$$v_{cm} = \omega \cdot R$$



Disk rolling on  
surface : translation  
of cm + rotation  
w.r.t center axis

$$KE = \frac{1}{2}mv_{cm}^2 + \frac{1}{2}I\omega^2$$

↑  
related!

$$= \frac{1}{2}mV_{cm}^2 + \frac{1}{2}\left(\frac{1}{2}mR^2\right)\left(\frac{V_{cm}}{R}\right)^2$$

$$= \frac{1}{2}mV_{cm}^2 + \frac{1}{4}mV_{cm}^2$$

$$= \frac{1}{2}\left(\frac{3}{2}m\right)V_{cm}^2$$



→ Consequence of rolling vs skidding: 50% increase in mass  
(ABS braking system → car loses KE in shorter distance).

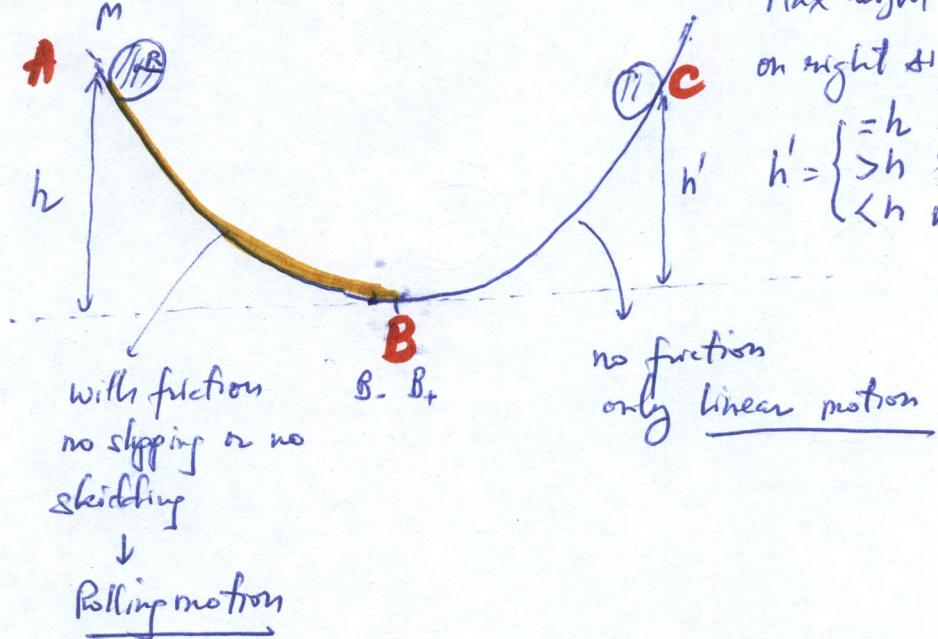
↳ Anti-locking braking system

$\left\{ \begin{array}{l} \text{w/o ABS: inertia mass} \\ \text{is } 1 \text{ m} \\ \text{w ABS: inertia mass} \\ \text{is } 1.5 \text{ m} \end{array} \right.$

↳ Shorter stopping  
distance.

10.64

Working with both rolling motion (left side) and linear motion (right side)



Statement: 1) There is conservation of energy b/w B & C.

2) Friction allows relation b/w A & B : with the additional rotational motion we can establish conservation of energy b/w A & B as well!

$$\begin{array}{ccc} \text{initial (A)} & = & \text{final (B)} \\ \textcircled{1} \quad Mg h & = & \frac{1}{2} M v_B^2 + \frac{1}{2} I \omega^2 \end{array} \quad \begin{array}{ccc} \text{initial (B)} & = & \text{final (C)} \\ \textcircled{2} \quad \frac{1}{2} M v_B^2 & = & Mg h' \end{array}$$

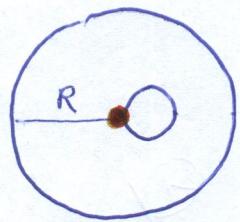
→ Find  $v_B$  then  $h'$

$$\left\{ \begin{array}{l} \text{Rolling motion: } v_B = \omega \cdot R \Rightarrow \omega = \frac{v_B}{R} \\ \text{Sphere's moment of inertia: } I = \frac{2}{5} MR^2 \end{array} \right\} \quad \begin{aligned} \textcircled{1} \rightarrow Mg h &= \frac{1}{2} M v_B^2 + \frac{1}{2} \left( \frac{2}{5} MR^2 \right) \left( \frac{v_B}{R} \right)^2 \\ &= \frac{1}{2} M v_B^2 + \frac{1}{2} \left( \frac{2}{5} M \right) v_B^2 \\ &= \frac{1}{2} \left( \frac{7}{5} M \right) v_B^2 \end{aligned}$$

$$\Rightarrow \frac{1}{2} M v_B^2 = \frac{5}{7} M g h$$

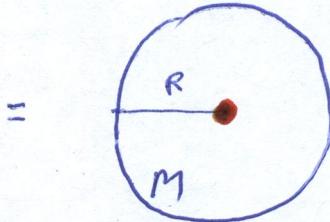
$$\textcircled{2} \rightarrow \frac{5}{7} M g h = M g h' \Rightarrow h' = \frac{5}{7} h$$

10.65



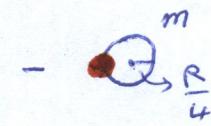
Disk with hole  
of radius  $\frac{R}{4}$

Center axis of rotation



Whole disk

Center axis of rotation



missing piece  
of radius  $\frac{R}{4}$

Tangential axis  
of rotation

Parallel axis  
Theorem

$$\boxed{I_{\text{with hole}}}$$

?

$$\underbrace{I_{\text{without hole}}}_{\frac{1}{2}MR^2}$$

$$\underbrace{I_{\text{missing piece}}}_{\frac{1}{2}m\left(\frac{R}{4}\right)^2 + m\left(\frac{R}{4}\right)^2}$$

$$m \text{ is a fraction of } M: \quad \frac{m}{M} = \frac{\pi\left(\frac{R}{4}\right)^2}{\pi R^2} = \frac{1}{16} \Rightarrow m = \frac{M}{16}$$

$$I_{\text{with hole}} = \frac{1}{2}MR^2 - \frac{3}{2}m\left(\frac{R}{4}\right)^2 = \frac{1}{2}MR^2 - \frac{3}{2}\frac{M}{16}\frac{R^2}{16}$$

$$= \frac{1}{2}MR^2 \left(1 - \frac{3}{16^2}\right) \\ \frac{253}{256}$$

## Ch 11 Rotational Vectors & Angular Momentum

$$\vec{F}_{\text{net}} = \frac{d\vec{p}}{dt} \leftrightarrow \vec{F}_{\text{net}} = 0 \text{ (collisions)} \leftrightarrow \vec{p}_i = \vec{p}_f \quad (\text{linear momentum is conserved})$$

$$\vec{\tau}_{\text{net}} = I\alpha \leftrightarrow \frac{d\vec{L}}{dt} \leftrightarrow \vec{\tau}_{\text{net}} = 0 \leftrightarrow \vec{L}_i = \vec{L}_f \quad (\text{angular momentum is conserved})$$

So far:  $r$  showed up  
in  $\vec{\tau}$  &  $I$

linear

$$\vec{p} = m\vec{v}$$

linear momentum

$\vec{L}$ : angular momentum  
(related to  $\vec{p}$  & radius does matter)

Rotational

$$\vec{L} = \vec{r} \times \vec{p}$$

angular momentum  
of an object wrt an axis  
of rotation equals the  
cross product b/w its  
position vector  $\vec{r}$  (wrt axis  
of rotation) and its linear  
momentum  $\vec{p}$

( $\vec{L}$  is perpendicular to  
both  $\vec{r}$  &  $\vec{p}$ )

1) So far:  $\left\{ \begin{array}{l} \vec{\tau} = \vec{r} \times \vec{F} \\ \vec{L} = \vec{r} \times \vec{p} \end{array} \right.$

$$\text{also } \vec{F} = \frac{d\vec{p}}{dt}$$

$$\boxed{\vec{\tau} = \frac{d\vec{L}}{dt}}$$

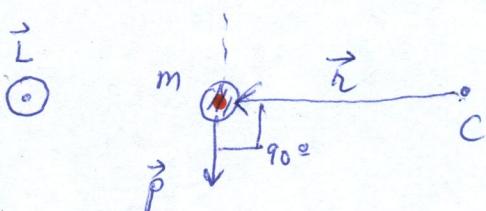
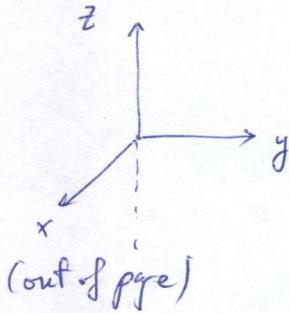
2)  $\boxed{\vec{\tau} = I \cdot \vec{\alpha}}$        $(\vec{F} = m\vec{a})$

$$\boxed{\vec{L} = I \cdot \vec{\omega}}$$
  

(recall  $\vec{\alpha} = \frac{d\vec{\omega}}{dt}$ )

Angular momentum =  $\begin{cases} \text{General motion: } \vec{L} = \vec{r} \times \vec{p} \\ \text{Rotations: } \vec{L} = I\vec{\omega} \end{cases}$

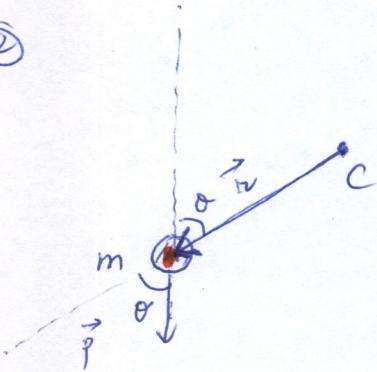
① → An object of mass  $m$  moving along  $-z$  axis ( $-\hat{k}$ ) ; our axis of "rotation" is @ C



Although there is no rotation we can calculate the angular momentum  $\vec{L} = \vec{r} \times \vec{p} = rpsin\theta \hat{i}$   
wrt to C

RHR

②

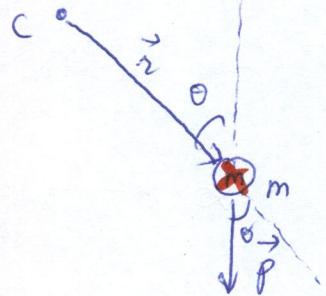


$$\vec{L} = rpsin\theta \hat{i} \quad (\text{out of page})$$

RHR

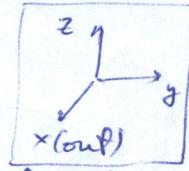
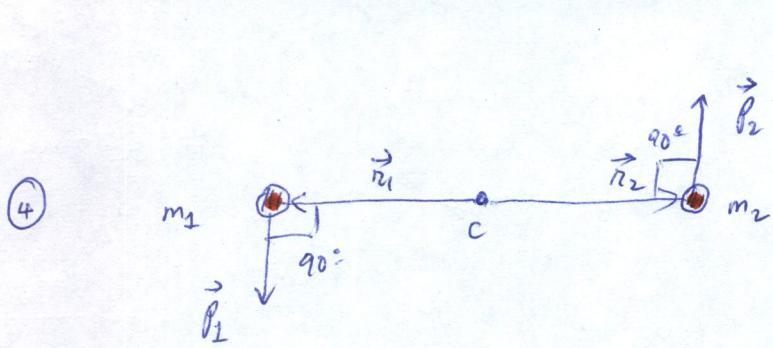
RHR: to find direction of  $\vec{r} \times \vec{p}$ : RH fingers along 1st vector; then close R&H toward 2nd vector, thumb gives direction of  $\vec{r} \times \vec{p}$

③



$$\vec{L} = rpsin\theta (-\hat{i})$$

RHR

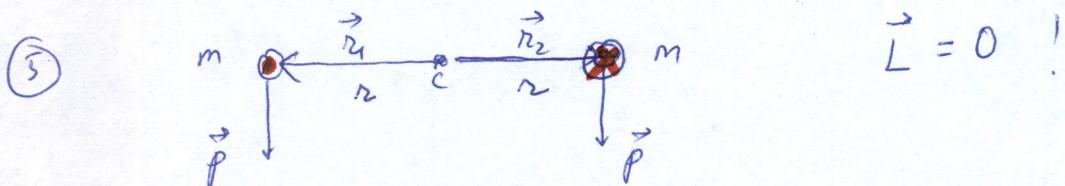


Angular momentum for this system?

$$\begin{aligned}\vec{L} &= \vec{r}_1 \times \vec{p}_1 + \vec{r}_2 \times \vec{p}_2 \\ &= r_1 p_1 \hat{i} + r_2 p_2 \hat{i} = (r_1 p_1 + r_2 p_2) \hat{i} \quad (\text{out of page!})\end{aligned}$$

(equal masses  $m_1 = m_2 = m$ ; equal radii:  $|\vec{r}_1| = |\vec{r}_2| = R$ ; equal speeds  $v_1 = v_2 = v$ )

$$\vec{L} = 2mrp \hat{i} \quad \text{⑤}$$



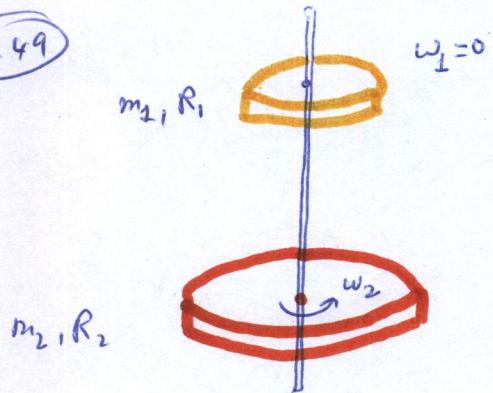
$$\vec{L} = 0 !$$

Conservation of angular momentum:

General motion:  $\vec{L}_i = \vec{r}_i \times \vec{p}_i = \vec{L}_f = \vec{r}_f \times \vec{p}_f$

Rotational motion  $I_i \omega_i = I_f \omega_f$

11.49



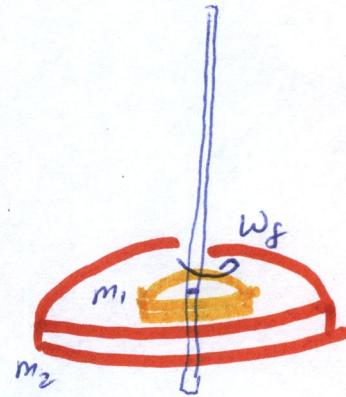
Top @ rest (dragging freely)  
Bottom @  $\omega_2 = 180 \text{ rpm}$

$$m_1 = 0.27 \text{ kg}$$

$$R_1 = 0.023 \text{ m}$$

$$m_2 = 0.44 \text{ kg}$$

$$R_2 = 0.035 \text{ m}$$



Two disks rotating together @  $\omega_f$

$= 180 \text{ rpm}$
$> 180 \text{ rpm}$
$< 180 \text{ rpm}$

Statement :  $\vec{\tau}_{\text{net}} = 0 \Leftrightarrow \vec{L}_i = \vec{L}_f$

Nothing is applied on either disk!

$$I_i \omega_i = I_f \omega_f$$

a)  $\boxed{\omega_f ?}$ 

$$\left\{ \begin{array}{l} I_1 = I_2 = \frac{1}{2} m_2 R_2^2 \\ \omega_1 = \omega_2 \\ I_f = I_1 + I_2 = \frac{1}{2} m_1 R_1^2 + \frac{1}{2} m_2 R_2^2 \end{array} \right.$$

$$\Rightarrow I_2 \omega_2 = (I_1 + I_2) \omega_f$$

$$\omega_f = \frac{I_2}{I_1 + I_2} \omega_2$$

$$= \frac{m_2 R_2^2}{m_1 R_1^2 + m_2 R_2^2} \omega_2$$

$$= \frac{0.44 \times 0.035^2}{0.27 \times 0.023^2 + 0.44 \times 0.035^2} \cdot 180 \text{ rpm}$$

$$\boxed{\omega_f = 142 \text{ rpm}}$$

b) Fraction of energy lost to friction (b/w top disk & bottom disk !)

$$\frac{KE_i - KE_f}{KE_i}$$

$$\frac{KE_i - KE_f}{KE_i} = 1 - \frac{KE_f}{KE_i} = 1 - \frac{\frac{1}{2} (I_1 + I_2) \omega_f^2}{\frac{1}{2} I_2 \omega_2^2} = 1 - \frac{0.27 \times 0.023^2 + 0.44 \times 0.035^2}{0.44 \times 0.035^2} \cdot \frac{142^2}{180^2}$$

$$= 0.21 \text{ or } 21\%$$

(note conversion from rpm to rad would cancel out up & down)

10.36

Rotational vs. translational kinetic energy

$$\text{Baseball} \leftrightarrow \text{sphere (solid)} \quad \left\{ \begin{array}{l} m = 0.15 \text{ kg} \\ R = 0.037 \text{ m} \end{array} \right\} \quad \left\{ \begin{array}{l} v_{cm} = 33 \text{ m/s} \\ \omega = 42 \text{ rad/s} \end{array} \right.$$

$$\rightarrow KE = \frac{1}{2}mv_{cm}^2 + \underbrace{\frac{1}{2}I\omega^2}_{\text{Rotational}} \Rightarrow f = \frac{\frac{1}{2}I\omega^2}{\frac{1}{2}mv_{cm}^2 + \frac{1}{2}I\omega^2} \quad \text{Rotational}$$

$$\qquad\qquad\qquad \text{Total KE}$$

Solid sphere

(2)

$$I = \frac{2}{5}mR^2 = \frac{2}{5}0.15 \cdot 0.037^2$$

$$f = \frac{\frac{1}{2} \cdot \frac{2}{5} 0.15 \times 0.037^2 \times 42^2}{\frac{1}{2} 0.15 \cdot 33^2 + \frac{1}{2} \cdot \frac{2}{5} 0.15 \times 0.037^2} = \frac{\frac{1}{2} \cdot 8 \cdot 2 \times 10^{-5} \times 42^2}{81 + \frac{1}{2} \cdot 8 \cdot 2 \times 10^{-5} \times 42^2} = 16$$

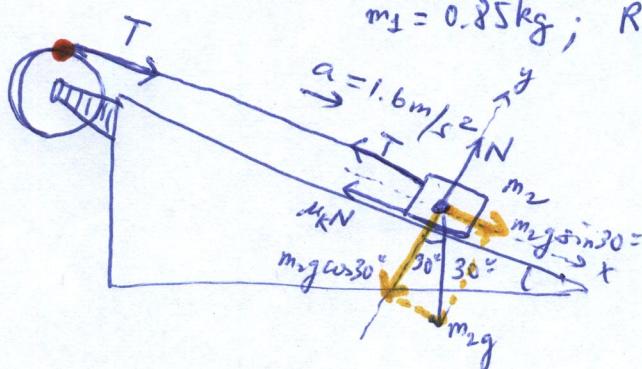
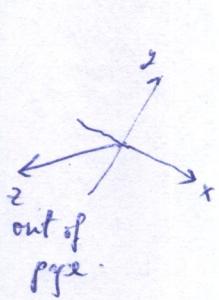
$$f = \frac{0.072}{81 + 0.072} = 8.88 \times 10^{-4}$$

10.57

Two objects

- (1) box under linear motion down a slope with friction  
 $m_2 = 2.4 \text{ kg}$   
 (2) drum under rotational motion  
 $m_1 = 0.85 \text{ kg}; R_1 = 0.05 \text{ m}$

→ String has negligible mass (same tension throughout)



$$Box: (F_{net} = ma)$$

$$F_{net_x} = m_2 g \sin 30^\circ - \cancel{\mu_k N} - \cancel{T} = m_2 a$$

$$F_{net_y} = N - m_2 g \cos 30^\circ = 0$$

$$T = m_2 g (\sin 30^\circ - \mu_k)$$

$$(T_{net} = I\alpha) \quad \text{Drum}$$



$$T_{net} = R_1 T \sin 90^\circ (-k) \quad \text{X}$$

$$R_1 T = \frac{1}{2} m_1 R_1 \alpha \Rightarrow T = \frac{1}{2} m_1 a$$

Statements i) Red dot has a linear acceleration same as that of the block (block + drum are connected by a rope)

$$\rightarrow \alpha = \frac{a}{R_1} \quad (\text{recall also: } \omega = \frac{v}{R}, \text{ since}$$

$$\theta = \frac{\text{arc}}{R} \rightarrow \frac{d\theta}{dt} = \frac{\text{(area)}}{R} \text{ or } \omega = \frac{v}{R})$$

Now using result from analysis for the drum rotational motion

$$T = \frac{1}{2} m_1 a$$

$$\rightarrow m_2 g \sin 30^\circ - \mu_k m_2 g \cos 30^\circ - \frac{1}{2} m_1 a = m_2 a$$

$$\mu_k = \frac{m_2 g \sin 30^\circ - (\frac{1}{2} m_1 + m_2) a}{m_2 g \cos 30^\circ}$$

$$= \frac{2.4 \times 9.81 \times \sin 30^\circ - \left(\frac{0.85}{2} + 2.4\right) 1.6}{2.4 \times 9.81 \times \cos 30^\circ}$$

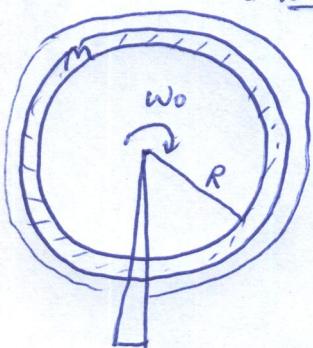
$$\mu_k = \underline{\underline{0.3557}}$$

10.58

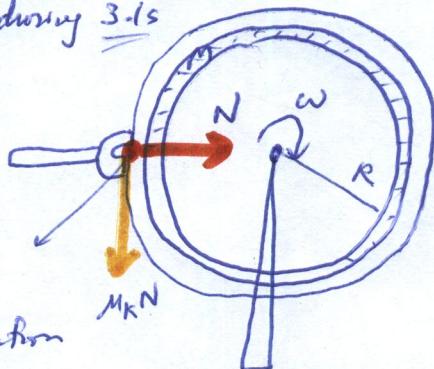
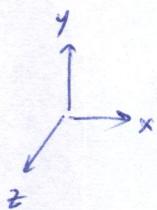
25

Spinning inverted like wheel  
then a wrench was applied with  
a normal force  $\underline{2.7N}$  during  $3\text{s}$

$$\left. \begin{array}{l} R = 0.33\text{m} \\ m = 1.9\text{kg} \end{array} \right| \mu_k = 0.46$$

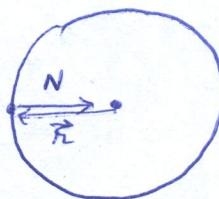


$$\omega_0 = 230\text{ rpm}$$



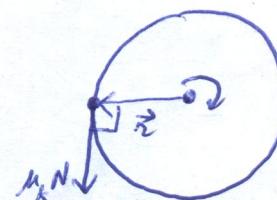
$$\omega < \omega_0$$

$\tau_{\mu_k N}$  applies an angular deceleration  
that will slow it down!



$$\vec{\tau}_N = \vec{r} \times \vec{N} = rN \underbrace{\sin 180^\circ}_{=0}$$

Normal force itself  
applies no torque



$$\vec{\tau}_{\mu_k N} = r\mu_k N \underbrace{\sin 90^\circ}_{=1} \vec{k}$$

Friction force applies  
torque  $\vec{\tau}_{\mu_k N} = R\mu_k N$   
(out of page)

Analog of 2nd Newton's law:

Bike wheel = a ring of  
mass  $m$  & radius  $R$

$$\tau_{\text{ext}} = I \cdot \alpha$$

$$\tau_{\mu_k N} = mR^2 \cdot \alpha$$

$$\alpha = \frac{\mu_k NR}{mR^2} \Rightarrow \boxed{\alpha = \frac{\mu_k N}{mR}}$$

Statement: how is it a deceleration in our equations?

Friction torque was out of page



$$\begin{matrix} \vec{\tau} \\ \vec{L} \end{matrix} \quad \begin{matrix} \vec{\tau}_{\mu_k} \\ \otimes \end{matrix} \quad \begin{matrix} \text{opposite} \\ \rightarrow \text{deceleration} \\ \rightarrow \alpha \text{ is negative} \end{matrix}$$

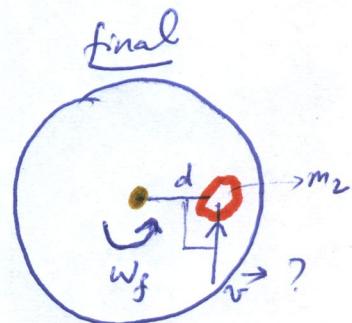
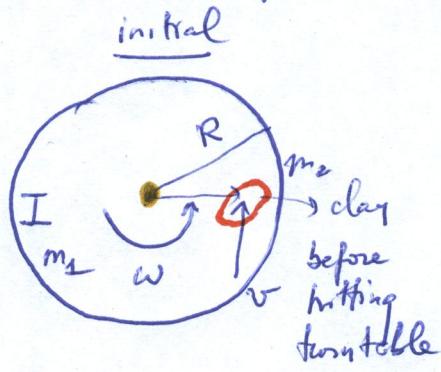
$$\left. \begin{array}{l} \vec{\tau} = \vec{r} \times \vec{F} \\ \vec{L} = \vec{r} \times \vec{p} \end{array} \right\} \Rightarrow \vec{\tau} = \frac{d\vec{L}}{dt}$$

$$= I\vec{\omega} \text{ (rotation)}$$

cw  $\vec{L}$  by RHR (fingers turning in cw, thumb into page)

11.45

Turn table (disk) rotating wrt its center axis, moment of inertia  $I$ . View from above:



$$\text{Find } v \quad \begin{cases} \text{a) } w_f = \frac{w}{2} \\ \text{b) } w_f = w \\ \text{c) } w_f = 2w \end{cases}$$

- clay hitting turntable horizontally & @  $90^\circ$  to radius in same direction as rotation.
- stays on with same final  $w_f$

Statement: to relate  $w_f$  to  $w$  (initial) :

$$\tau_{\text{net}} = 0 \text{ on turn table \& clay} \rightarrow \frac{dL}{dt} = 0 \rightarrow L_i = L_f$$

System is rotational motion  $\rightarrow L = I \cdot w$

$$L_i = L_f$$

$$\underbrace{\frac{1}{2}m_1R^2 \cdot w}_{\text{turntable}} + \underbrace{dm_2v}_{\text{clay}} = \left( \frac{1}{2}m_1R^2 + m_2d^2 \right) \cdot w_f$$

(there is angular momentum although no rotation as yet)

$$L = \vec{r} \times \vec{p}$$

( $\vec{p} \perp \vec{r}$  as solid

$$\theta = 90^\circ$$

$$\text{Solv for } v = \frac{\left( \frac{1}{2}m_1R^2 + m_2d^2 \right) w_f - \frac{1}{2}m_1R^2 w}{m_2 d}$$

a) if  $\omega_f = \frac{\omega}{2}$   $\rightarrow v = \frac{\frac{1}{2}m_1R^2\omega - \frac{1}{2}m_1R^2\bar{\omega} + m_2d^2\frac{\omega}{2}}{m_2d}$

$$= \frac{\frac{1}{2}m_2d^2\omega - \frac{1}{4}m_1R^2\bar{\omega}}{m_2d} = \frac{1}{2}dw - \frac{1}{4}\frac{m_1R^2}{m_2d}\omega$$

$$= \left(\frac{1}{2}d - \frac{1}{4}\frac{m_1R^2}{m_2d}\right)\omega$$

b) if  $\omega_f = \omega$   $v = \frac{m_2d^2\omega}{m_2d} = dw$

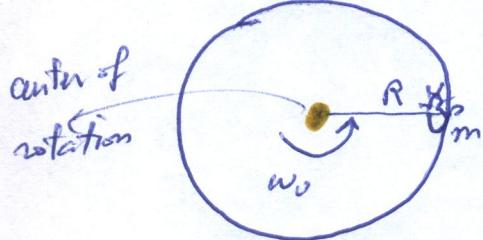
c) if  $\omega_f = 2\omega$   $v = \frac{m_1R^2\omega - \frac{1}{2}m_1R^2\bar{\omega} + 2m_2d^2\omega}{m_2d}$

$$= \frac{\frac{1}{2}m_1R^2\bar{\omega} + 2m_2d^2\omega}{m_2d}$$

$$= \left(\frac{1}{2}\frac{m_1R^2}{m_2d} + 2dw\right)\omega$$

11.38

→ Mouse walks on a turn table ( $I, R$  given) from outer edge to center. View from above:



$\omega_0 = 22\text{ rpm}$   
freely (no friction)



$\omega > \omega_0$

a)  $\tau_{\text{net}} = 0 \rightarrow L_i = L_f \quad (\text{rotations: } L = I \cdot \omega)$   
on system of disk & mouse

$$I \cdot \omega_0 + mR^2\omega_0 = I\omega \rightarrow \text{solve for } \omega$$

$$\text{1st kinematic of for rotational motion: } \omega = \omega_0 - \frac{\alpha \cdot t}{\Delta \omega}$$

230 rpm

non SI  
unit

Note: if we use SI units  $\rightarrow \alpha$  comes out as  $\frac{\text{rad}}{\text{s}^2}$  or  $\alpha \cdot t$  as  $\frac{\text{rad}}{\text{s}}$   
 $\rightarrow$  We will calculate  $\Delta \omega = \alpha \cdot t$  in  $\frac{\text{rad}}{\text{s}}$  then convert it to rpm before the subtraction

$$\Delta \omega = \frac{\mu_k N}{m R} \cdot t = \frac{0.46 \times 2.7}{1.9 \times 0.33} \times 3.1 \cdot \frac{\text{rad}}{\text{s}} \cdot \frac{60 \text{ s}}{\text{min}} \cdot \frac{1 \text{ rev}}{2\pi \text{ rad}} = 58.6 \frac{\text{rev}}{\text{min}}$$

$$\Delta \omega = 58.6 \text{ rpm} \Rightarrow \omega = 230 - 58.6 = 171 \text{ rpm}$$

Statements :

What are the forces acting on the wheel? Is  $\mu_k N R$  the net torque  $\tau_{\text{net}}$  on wheel?

Forces:

$N$  (by wrench),  $\mu_k N$  (friction),  $mg$  (weight),  $N'$  (normal by support on wheel)

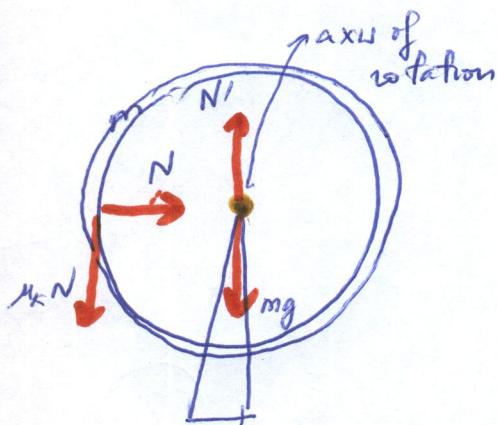
Torques:

$O$  ( $\vec{N} + \vec{r}$  forms angle  $180^\circ$ )

$\mu_k N R$

$O$  ( $\vec{r} = 0$  since force application point on axis of rotation)

$O$  (some reason as no torque for  $mg$ /



$$\omega = \frac{(I + mR^2)}{I} \omega_0 = (1 + \frac{mR^2}{I}) \omega_0$$

$$= \left(1 + \frac{0.0195 \times 0.25^2}{0.0154}\right) 22 \text{ rpm} = \frac{23.7411}{22.1714} \text{ rpm}$$

b) Work done by mouse : since system speeds up by  $0.1714 \text{ rpm}$

as it walks from outer edge  
to center of rotation

Should equal  $KE_f - KE_i = \frac{1}{2} I \omega_f^2 - \frac{1}{2} (I + mR^2) \omega_0^2$

$$= \frac{1}{2} \times 0.0154 \times 2.4868^2 - \frac{1}{2} (0.0154 + 0.0195 \times 0.25^2)$$

✓  
Need  $\omega$ 's in  $\frac{\text{rad}}{\text{s}}$

$$= \boxed{\frac{0.0035}{0.0026} \cdot 2.3038^2}$$

$$\omega_0 = 22 \frac{\text{rev}}{\text{min}} \cdot \frac{1 \text{ min}}{60 \text{ s}} \cdot \frac{2\pi \text{ rad}}{\text{rev}} = \frac{22 \times 2\pi}{60} \frac{\text{rad}}{\text{s}} = 2.3038 \frac{\text{rad}}{\text{s}}$$

$$\omega_f = 23.7414 \times \frac{2\pi}{60} \frac{\text{rad}}{\text{s}} = 2.4868 \frac{\text{rad}}{\text{s}}$$

(11.35)

Rotor:

$$R = \frac{d}{2} = 150 \times 10^{-6} \text{ m} \quad \left. \begin{array}{l} V_{oP} = \pi R^2 h \\ = \pi \times (150 \times 10^{-6})^2 \times 2 \times 10^{-6} \text{ m}^3 \end{array} \right\}$$

$$L = I \cdot \omega$$

$$= \frac{1}{2} m R^2 \cdot \omega$$

$$= \frac{1}{2} \rho V d R^2 \omega$$

$$= \frac{1}{2} 2329.6 \times \pi \times (1.5 \times 10^{-4})^2 \times 2 \times 10^{-6} \times (1.5 \times 10^{-4})^2 \times 800 \frac{2\pi}{60} = 3.1 \times 10^{-16} \frac{\text{kg m}^2}{\text{s}}$$

$$m = \rho \cdot V d \quad ; \quad \omega = 800 \frac{\text{rad}}{\text{min}} \cdot \frac{1 \text{ min}}{60 \text{ s}} \cdot \frac{2\pi}{\text{rad}}$$

↑ Silicon

$$= 800 \frac{2\pi}{60} \frac{\text{rad}}{\text{s}}$$

$$\rho = 2.3296 \frac{\text{g}}{\text{cm}^3} \cdot \frac{1 \text{ kg}}{1000 \text{ g}} \cdot \frac{10^6 \text{ cm}^3}{1 \text{ m}^3} = \boxed{2329.6 \frac{\text{kg}}{\text{m}^3}}$$